

# Lecture 4

## Heat engines

PHY 555 – Energy & Environment

Erik Johnson, Mathieu de Naurois, Daniel Suchet



# Lecture 4

## Heat engines

I. What, why, where ?

II. Thermodynamics framework

III. Standard cycles and their applications

IV. Perspectives



# Heat is work and work is heat – or is it ?



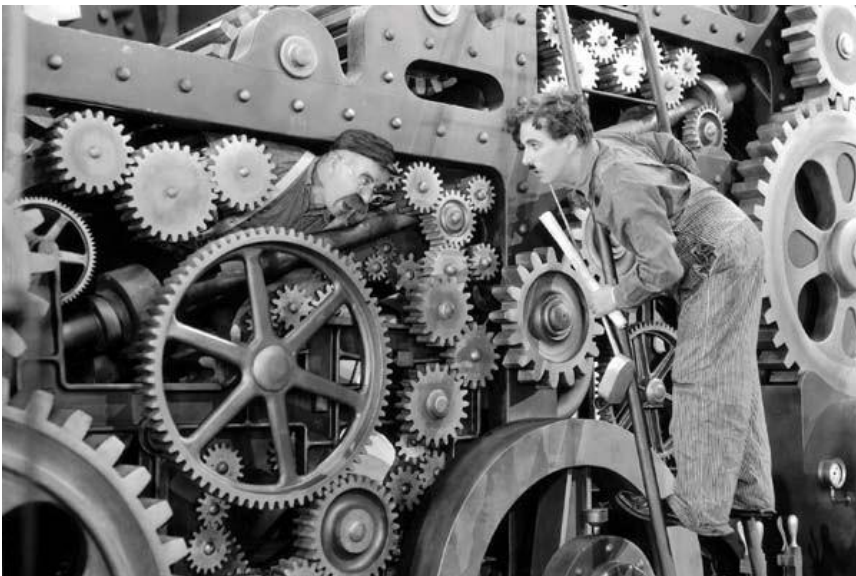
[https://www.youtube.com/watch?v=VnbiVw\\_1FNs](https://www.youtube.com/watch?v=VnbiVw_1FNs)

$$dU = \delta W + \delta Q$$

$$\delta W = \mathbf{F} \cdot d\mathbf{r} = -dE_p$$

(for conservative forces)

$$\delta Q = T \delta S_{\text{exchange}}$$



Hard to get

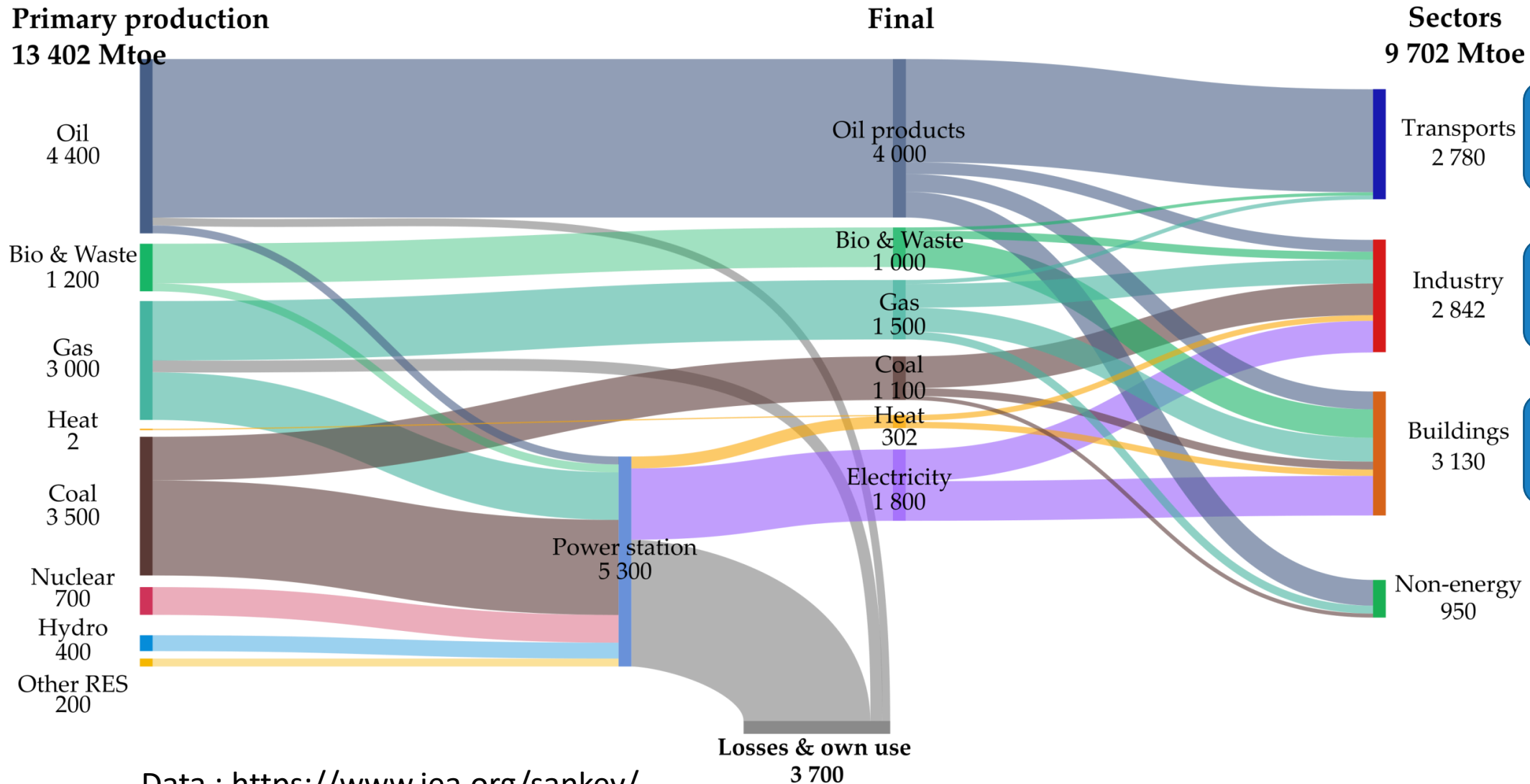
« Easy » to use



Easy to get (combustion)

Hard to use

# Work = 50% of energy usage

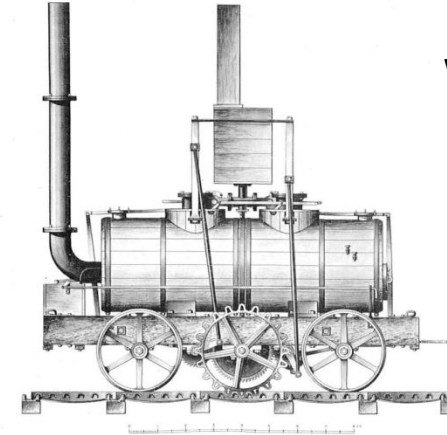


Data : <https://www.iea.org/sankey/>

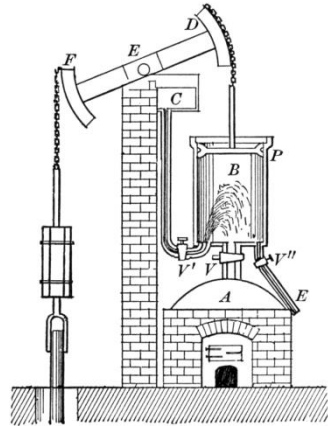
# A HEATED HISTORY



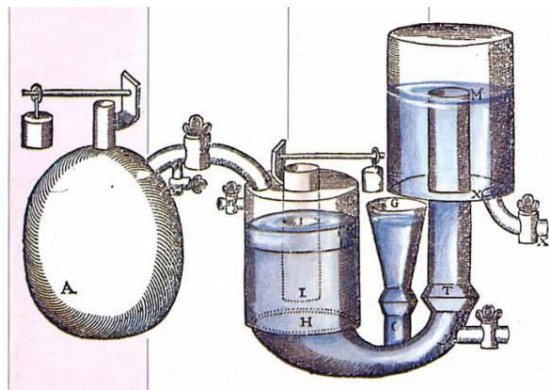
William Murdoch & Reynolds  
Matthew Murray  
Locomotive



Thomas Newcomen & Savery  
Self acting steam engine



90 1700 10 20 30 40 50 60 70 80 90 1800 10 20 30 40 50 60 70 80



Denis Papin

"Nouvelle méthode pour obtenir à bas prix des forces considérables" (1690)

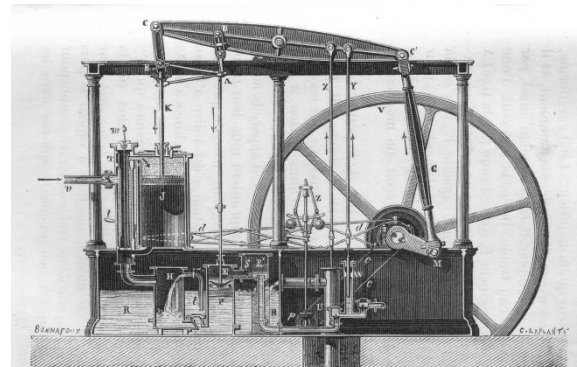
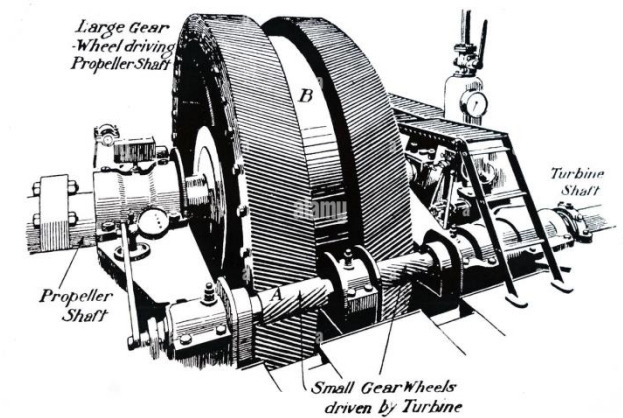


Fig. 59. — Machine à balancier de Watt.  
e. Tuyau de prise de vapeur; T, tiroir; J, cylindre; H, condenseur; PE pompe d'épuisement; WY pompe alimentaire de la chaudière; UX pompe d'alimentation de la bièche; R; Z régulateur; d' excentrique; ABCD parallélogramme; OM bielle et manivelle; V volant.

James Watt  
Improvements,  
Rotating motion

Charles Parson  
Turbine



# Turning heat into motion

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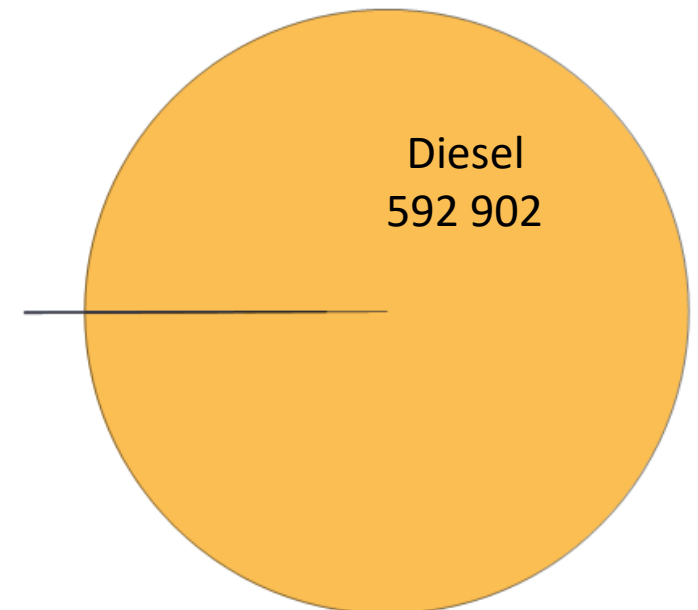
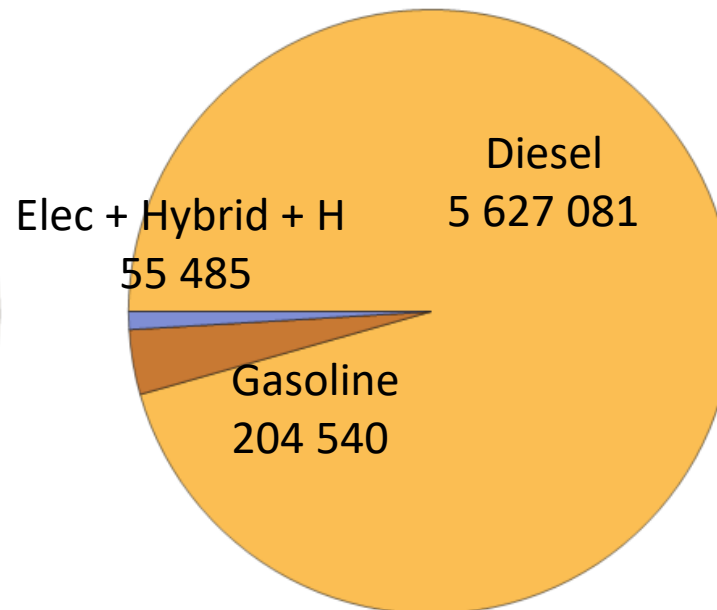
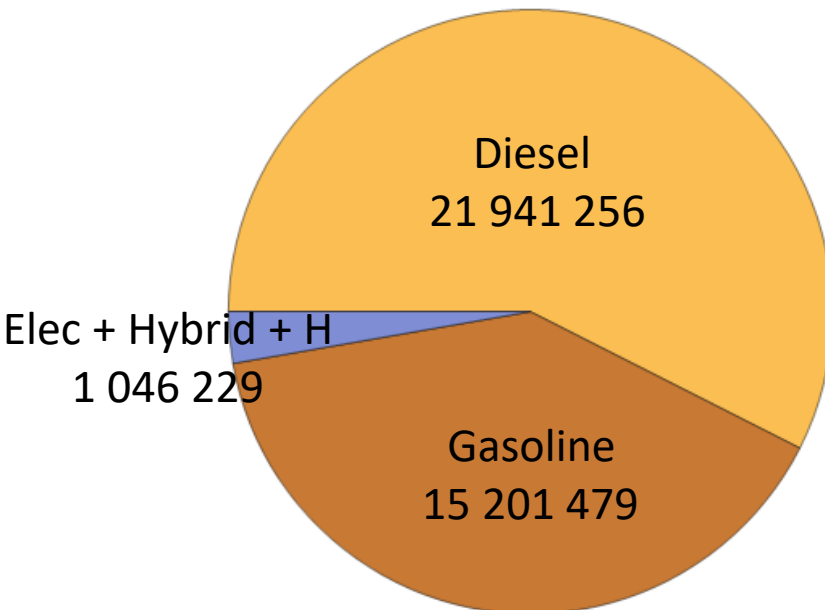


Vehicle fleet in France (2021).  
Data : SDES

Passenger car  
**38 346 266**

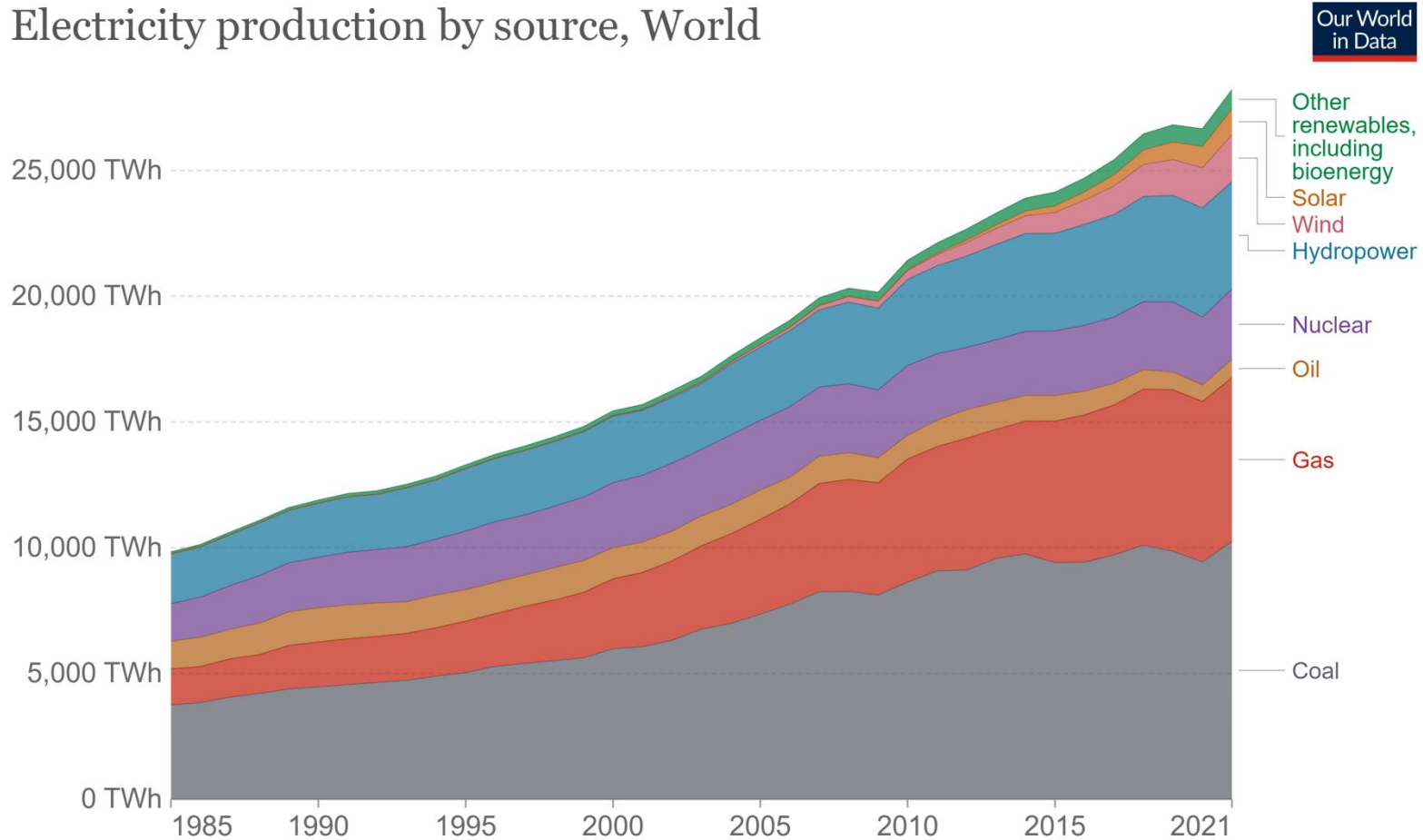
Light utility  
**5 904 396**

Heavy vehicles  
**600 283**

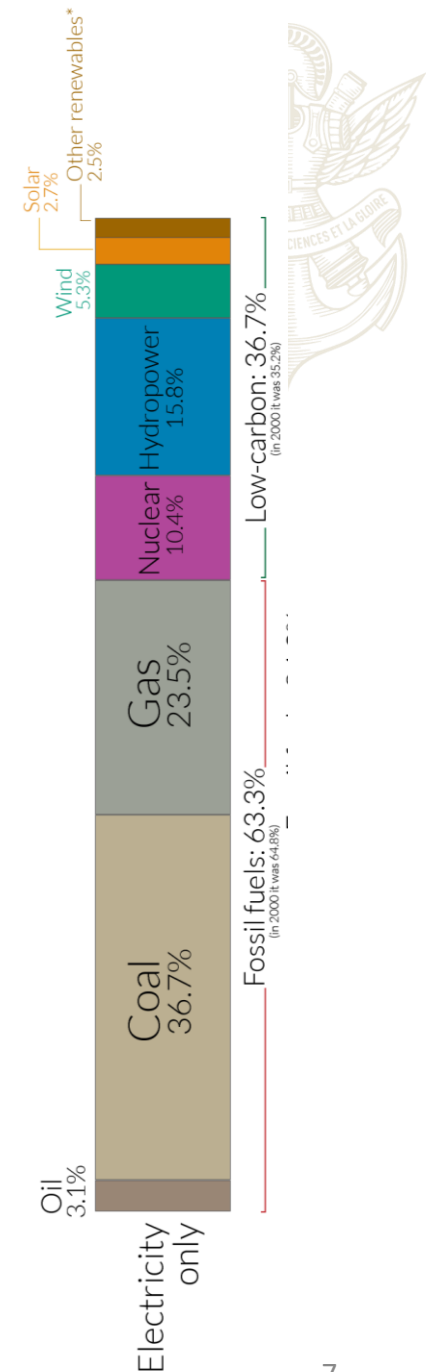


# Turning heat into electricity

Electricity production by source, World



Source: Our World in Data based on BP Statistical Review of World Energy (2022); Our World in Data based on Ember's Global Electricity Review (2022); Our World in Data based on Ember's European Electricity Review (2022)  
 Note: 'Other renewables' includes biomass and waste, geothermal, wave and tidal.  
 OurWorldInData.org/energy • CC BY

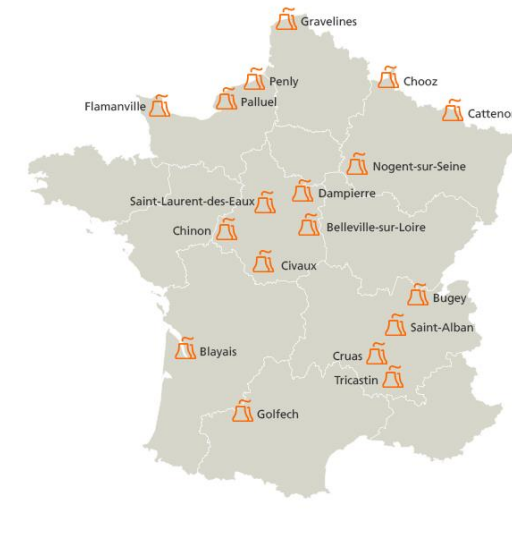


# French focus

Puissance	Nombre de réacteurs
1 450 MW	4
1 300 MW	20
900 MW	32

Nombre de réacteurs nucléaires en France par puissance

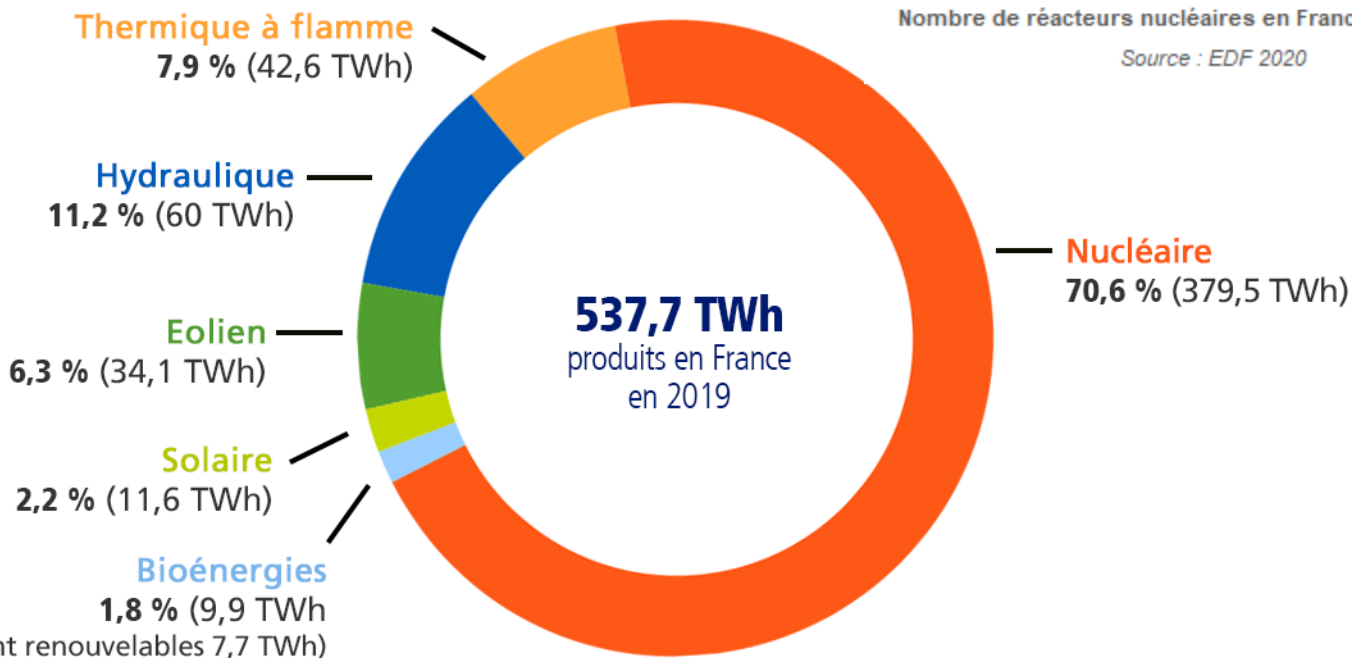
Source : EDF 2020



Répartition des centrales nucléaires en France en 2021

Source EDF 2021

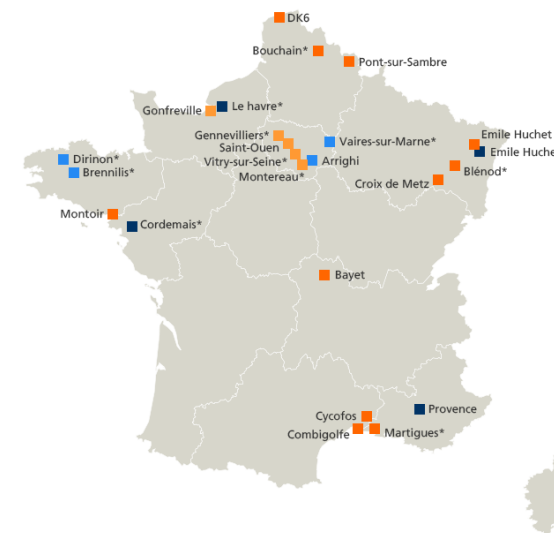
© EDF



La production française d'électricité en 2019

Source RTE - bilan électrique 2019

© EDF



CENTRALES À CHARBON ET À FIOUL

- Charbon
- Turbine à Combustion (TAC)

CENTRALES À GAZ

- Cycle Combiné Gaz (CCG)
- TAC Gaz

\* Centrale EDF

Répartition des centrales thermiques à flamme en France en 2021

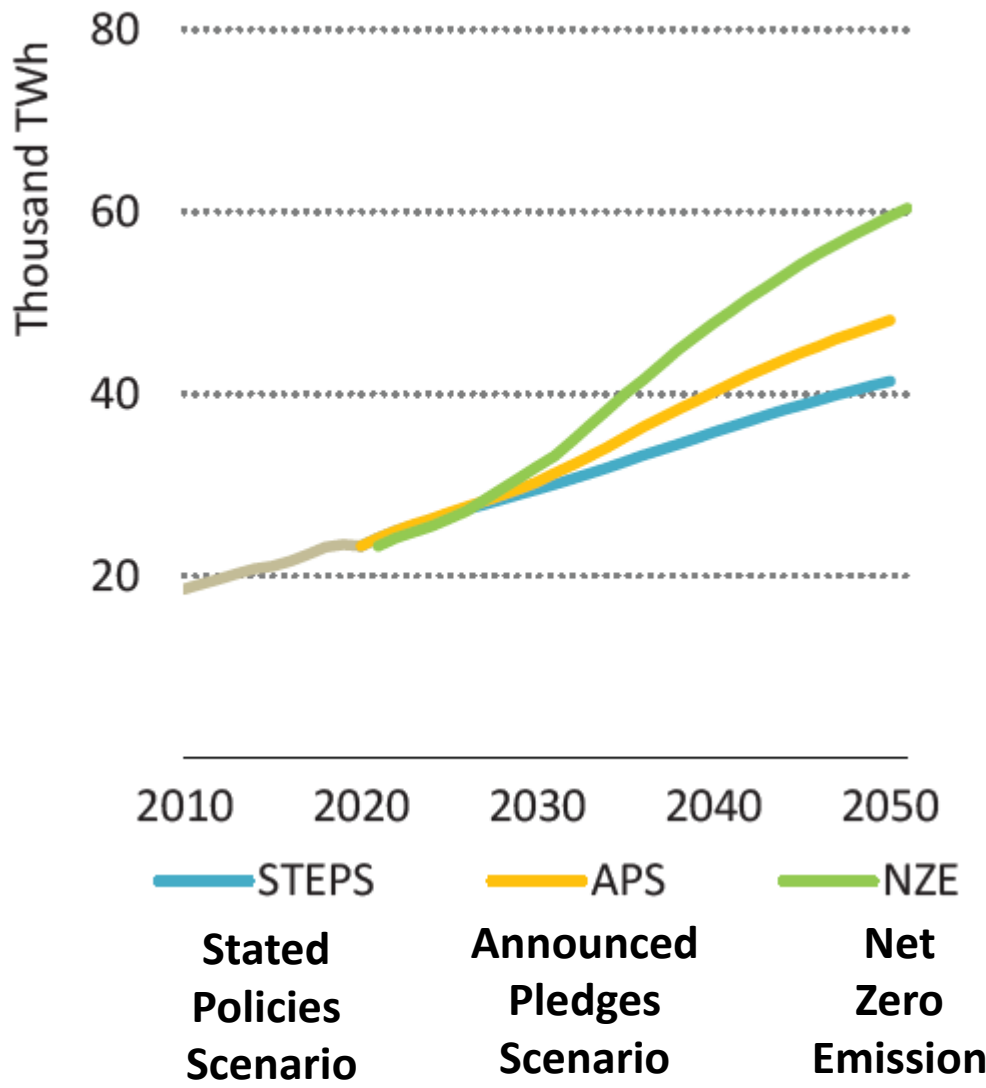
Source RTE - Bilan électrique 2019

© EDF





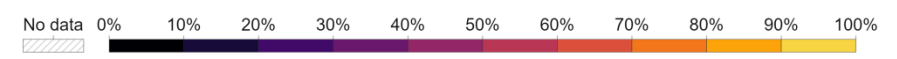
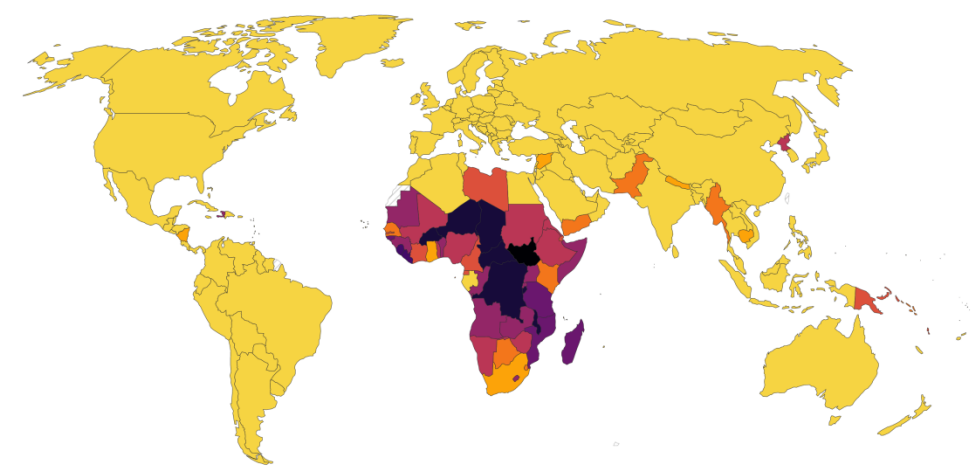
# Why electricity matters



10% of humanity doesn't have access to electricity

## Electricity access, 2020

Share of the population with access to electricity. The definition used in international statistics adopts a very low cutoff for what it means to 'have access to electricity'. It is defined as having an electricity source that can provide very basic lighting, and charge a phone or power a radio for 4 hours per day.



Source: World Bank

OurWorldInData.org/energy • CC BY

Electrification can be a tool towards decarbonization

# Lecture 4

## Heat engines

I. What, why, where ?

II. Thermodynamics framework

III. Standard cycles and their applications

IV. Perspectives

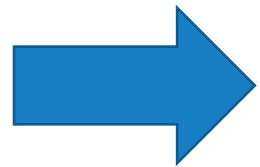


# Turning heat into work

Use a source to provide heat to the converter.

Heat comes with entropy.

Getting rid of entropy requires sacrificing some of the heat.



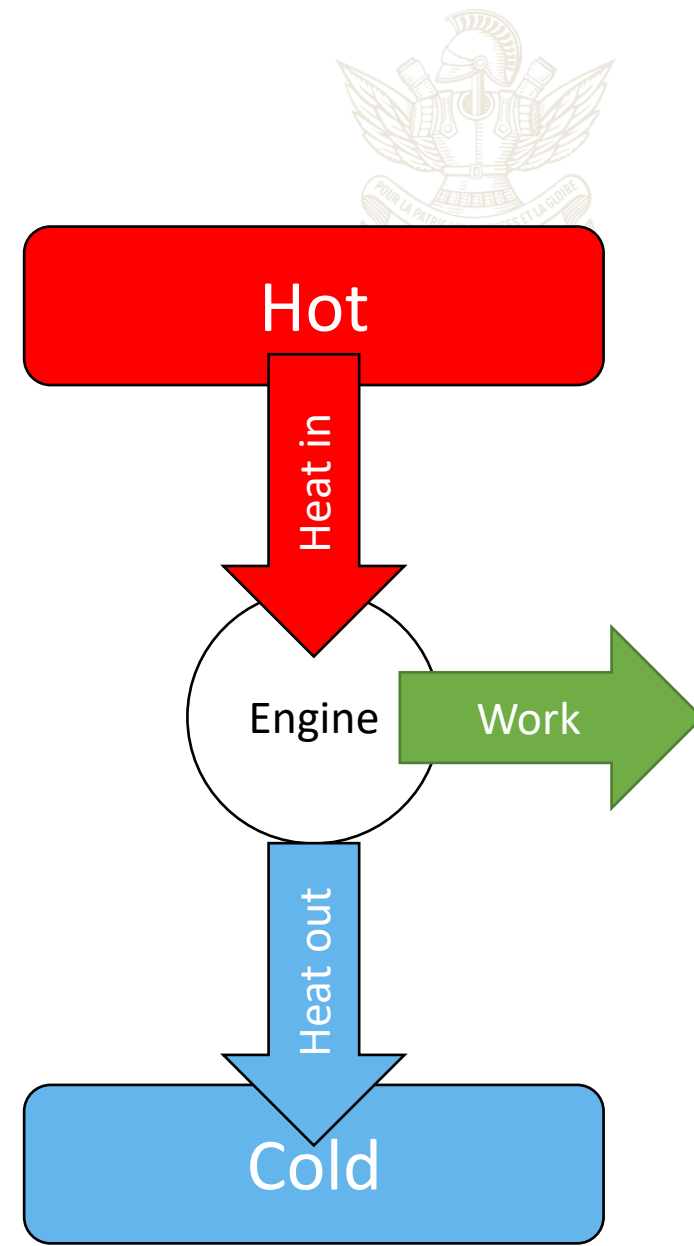
$$\text{work output} \leq \text{heat input} \left( 1 - \frac{T_{\text{cold}}}{T_{\text{hot}}} \right)$$

**Carnot efficiency**

Ultimate efficiency

Needs no details about operation

0 power !



# Turning heat into power

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The heat flow depends on the temperature difference

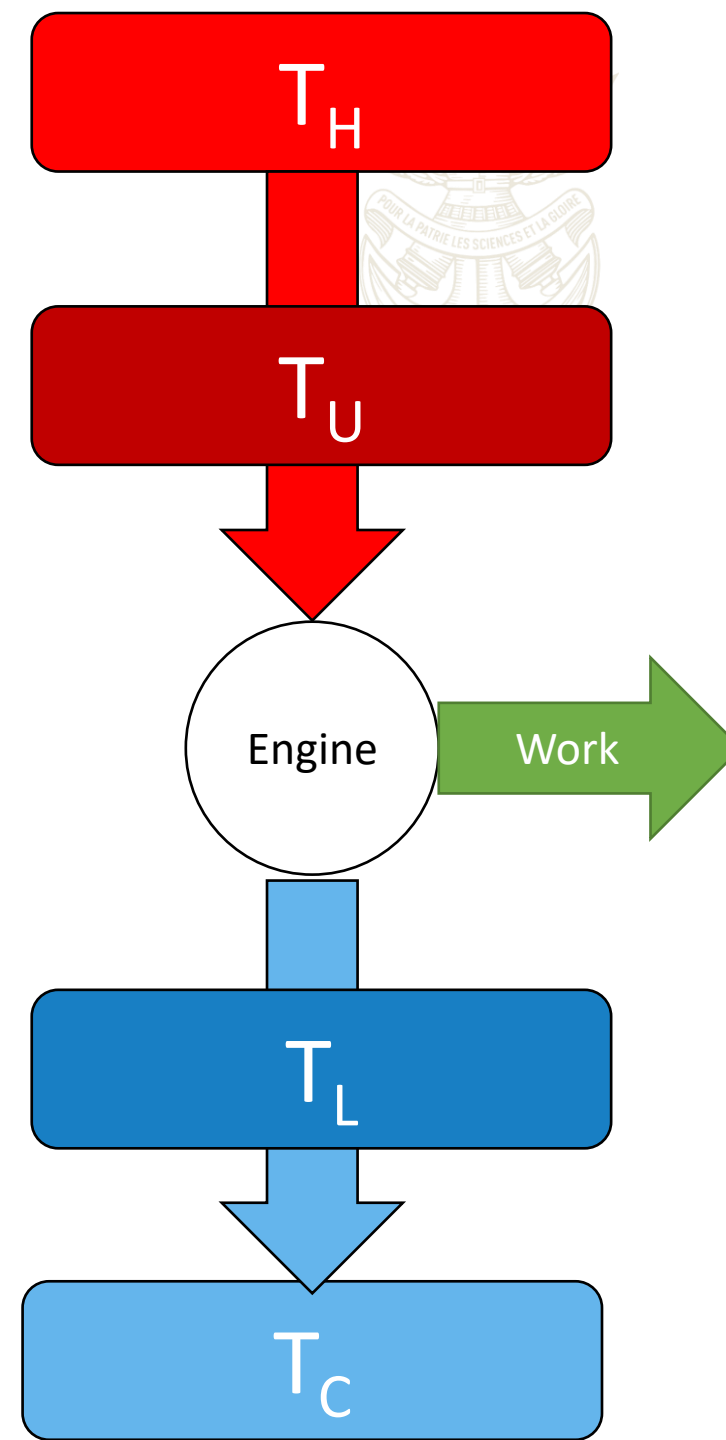
$$\dot{Q}_{\text{in}} \propto (T_H - T_U)$$

$$\dot{Q}_{\text{out}} \propto (T_C - T_L)$$

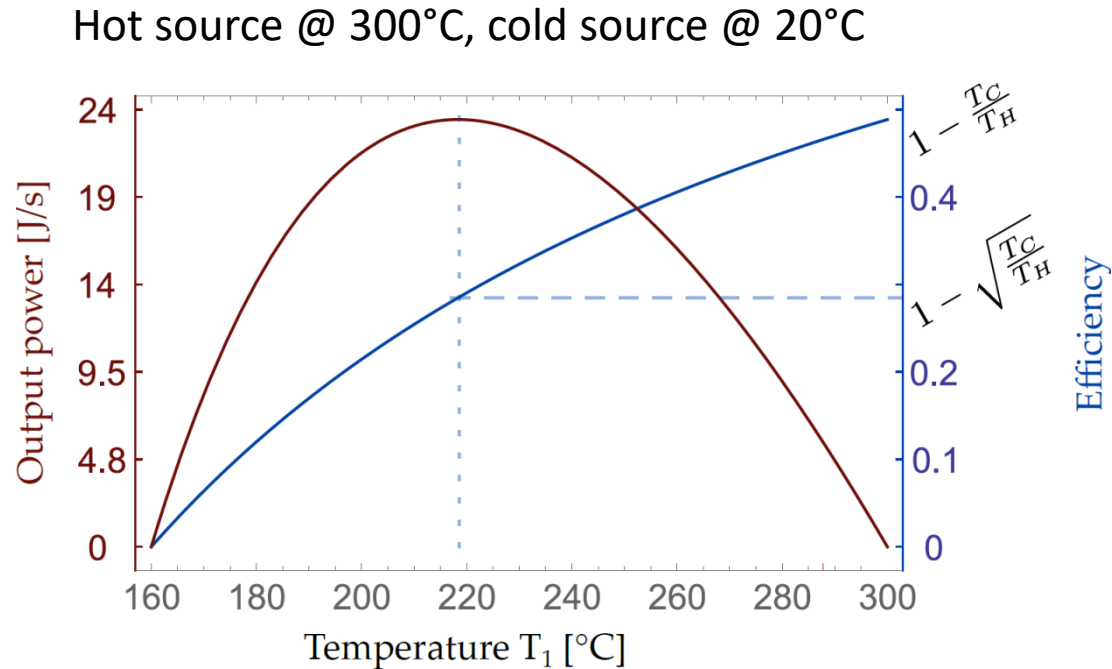
Larger temperature difference

- Larger heat flow 👍
- Larger entropy production 🙄

Any entropy input requires heat output

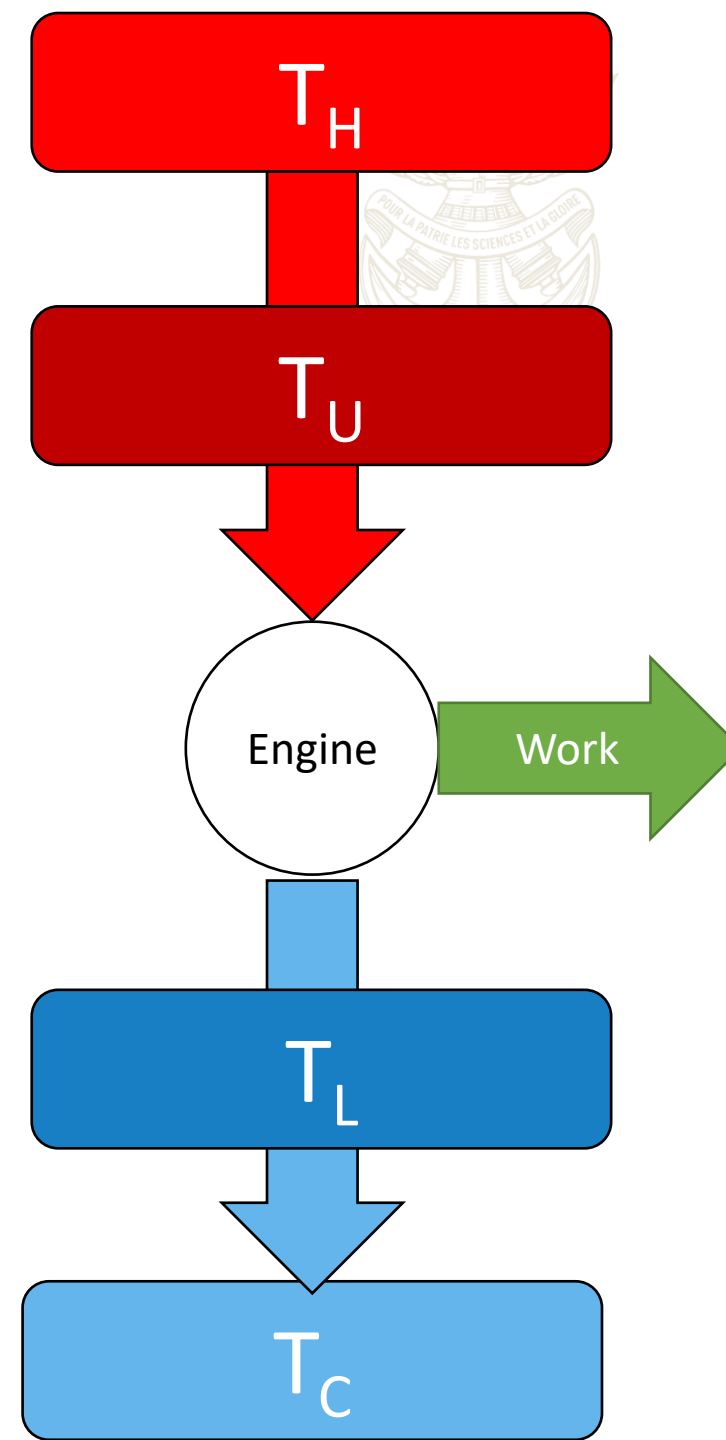


# Turning heat into power

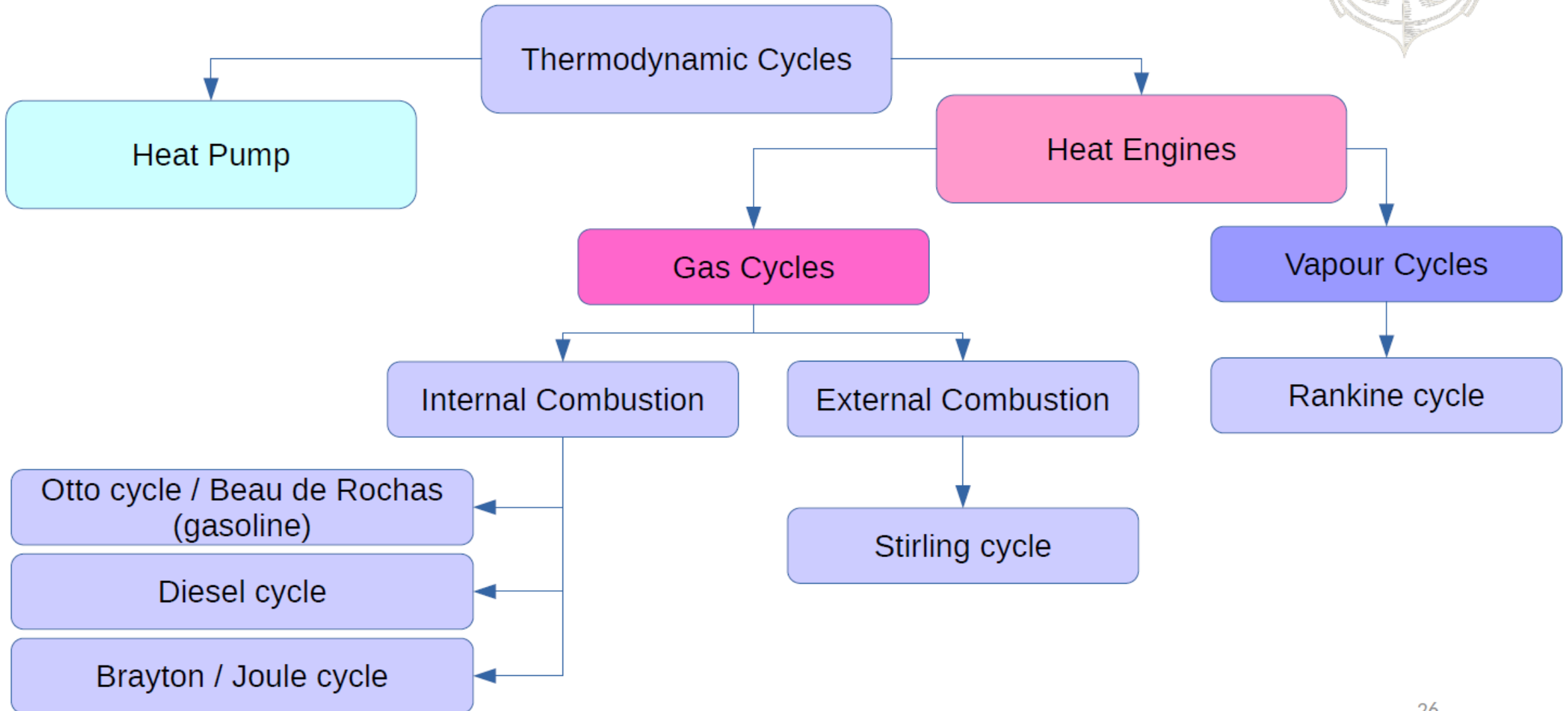


$$\text{work output} \leq \text{heat input} \left( 1 - \sqrt{\frac{T_{\text{cold}}}{T_{\text{hot}}}} \right)$$

Curzon-Ahlborn (or Chambadal-Novikov) efficiency



# The usual suspects



# Working principle



Pressure force :  $\mathbf{F} = -p_{\text{ext}}\mathbf{S}$

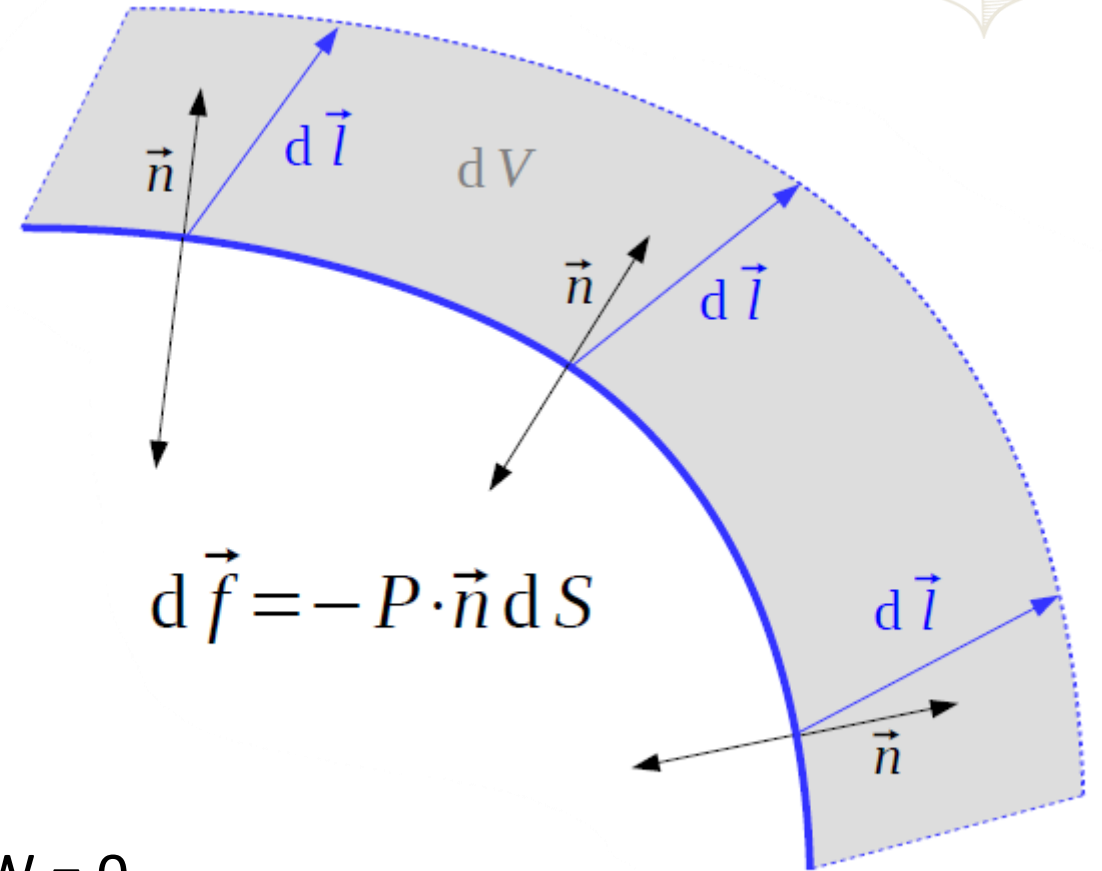
proportionnal to surface

$\perp$  to surface

1 bar =  $10^5$  N/m<sup>2</sup>

$$\delta W = \mathbf{F} \cdot d\mathbf{r} = -p_{\text{ext}} dV$$

Solid & liquid are almost incompressible  $\rightarrow W = 0$





# A word on enthalpy

Transformation under fixed external pressure

1st Law :

$$dU = \delta W_p + \delta W_{op} + \delta Q$$

Work of the ambient pressure :

$$\delta W_p = p_1 V_1 - p_2 V_2 = -d(pV)$$

Introducing enthalpy :

$$dH = d(U + pV) = \delta W_{op} + \delta Q$$

Fixed  $p_{ext}$

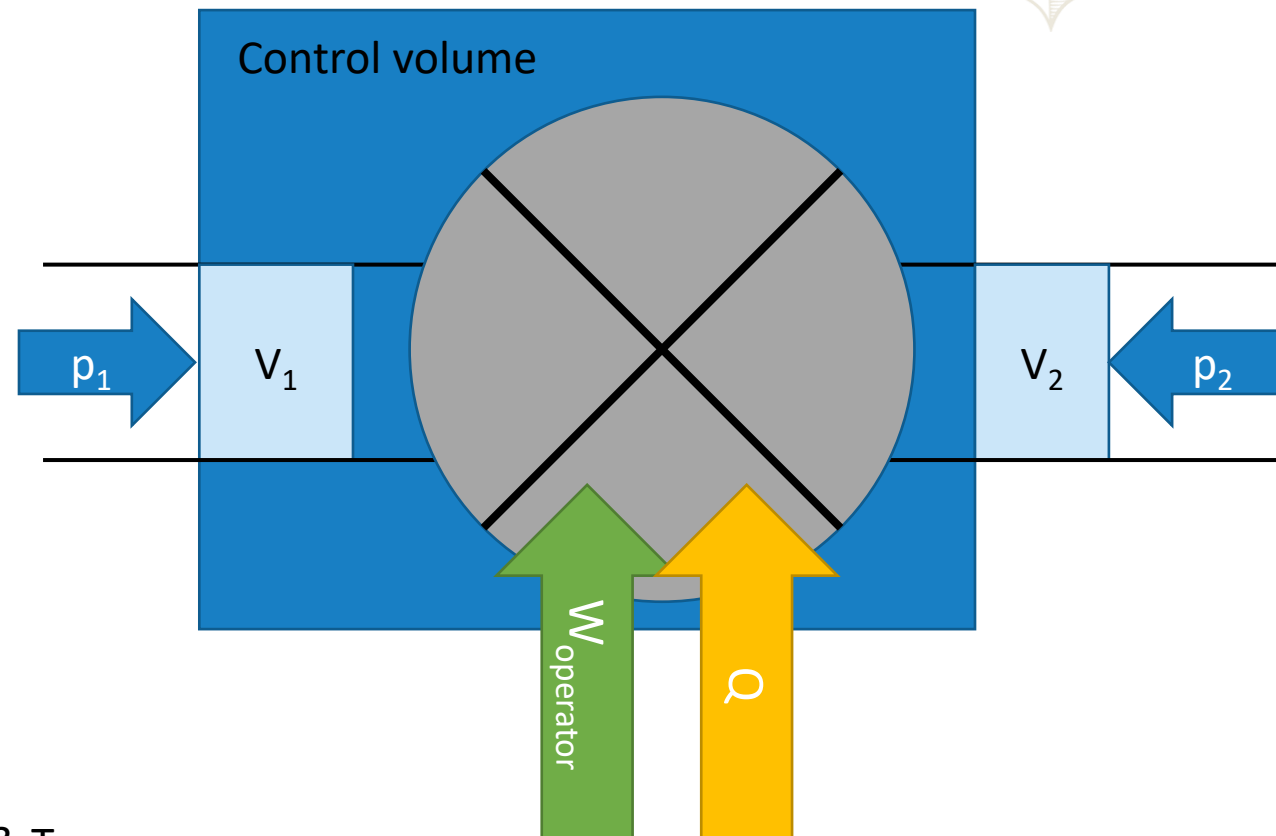
$$H = U + pV$$

Fixed  $T_{ext}$

$$F = U - TS$$

Fixed  $p_{ext}$  &  $T_{ext}$

$$G = U + pV - TS$$







# Everything you need to know about the ideal gas

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Equation of state

$$pV = nRT$$

$$R = k_B \mathcal{N}_A = 8.314 \text{ J/mol/K}$$

Joule's 1st law : internal energy depends only on temperature

$$dU = C_V dT$$

*Reminder :*

$$dU = -pdV + TdS + \mu dN$$

Joule's 2nd law : enthalpy depends only on temperature

$$dH = C_P dT$$

*Reminder :*

$$dH = Vdp + TdS + \mu dN$$

Mayer's relation

$$C_P - C_V = nR$$

Definition : Adiabatic index

$$\frac{C_P}{C_V} = \gamma = 1 + \frac{2}{f}$$

$$C_V = \frac{nR}{\gamma - 1}$$

$$C_P = \frac{\gamma nR}{\gamma - 1}$$

# Everything you need to know about the ideal gas



Equi-partition theorem

$$\frac{U}{N} = f \times \frac{1}{2} k_B T$$

Sackur Terode equation

$$\frac{S}{N} = k_B \left( \frac{5}{2} - \frac{\mu}{k_B T} \right)$$

Chemical potential

$$\mu = k_B T \ln \left( \left( \frac{h}{2\pi m k_B T} \right)^{3/2} \frac{N}{V} \right)$$

Entropy variation in an ideal gas:

$$\Delta S = n C_V \ln \left( \frac{p V^\gamma}{p_0 V_0^\gamma} \right) = n C_V \ln \left( \frac{T V^{\gamma-1}}{T_0 V_0^{\gamma-1}} \right) = n C_V \ln \left( \frac{T^\gamma p^{1-\gamma}}{T_0^\gamma p_0^{1-\gamma}} \right)$$

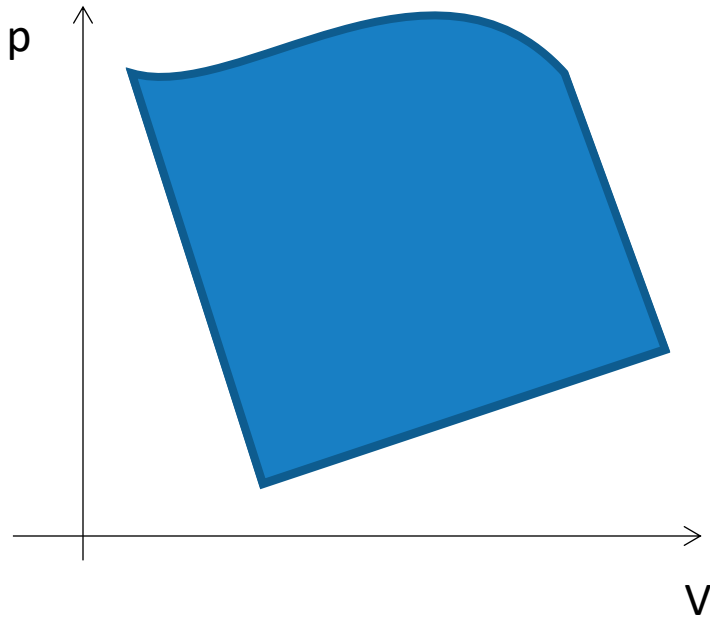
# Diagrams



pV diagram

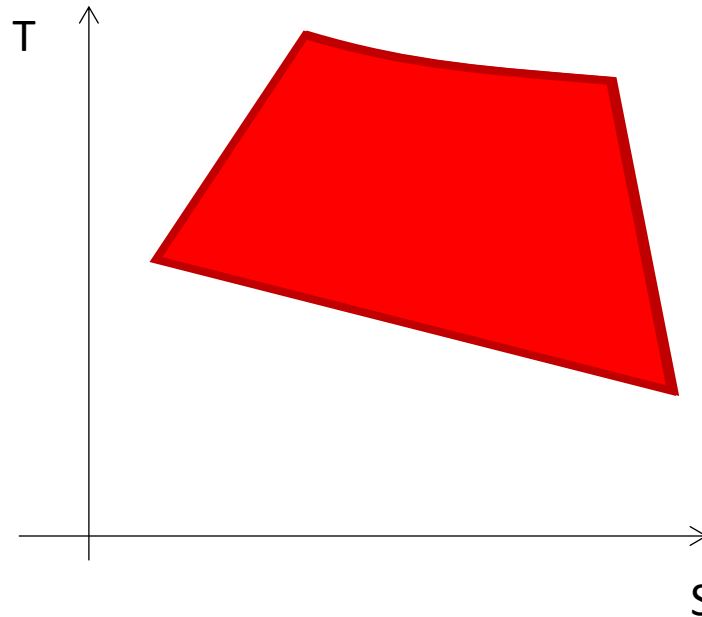
$$W = - \int p dV$$

<0 : engine  
>0 : heat pump

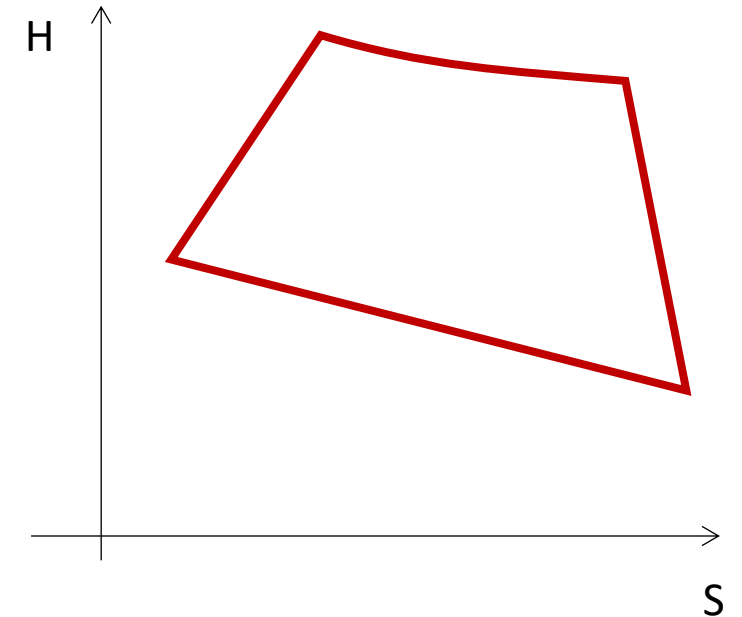


TS diagram

$$Q = \int T dS$$



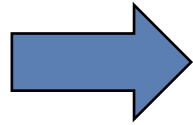
HS Molier diagram



# Usual transformations

**Isothermal**

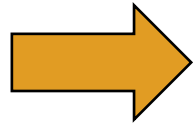
$$T = \text{cst}$$



$$pV = nRT = \text{cst}$$

**Adiabatic  
+ reversible**

$$\delta Q = 0$$

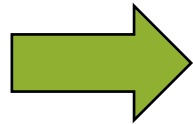


$$pV^\gamma = \text{cst}$$

$$TV^{\gamma-1} = \text{cst}$$

**Isobaric**

$$p = \text{cst}$$

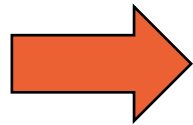


$$T_f = T_i \exp \Delta S / C_p$$

$$\Delta V = \frac{nR}{p} \Delta T$$

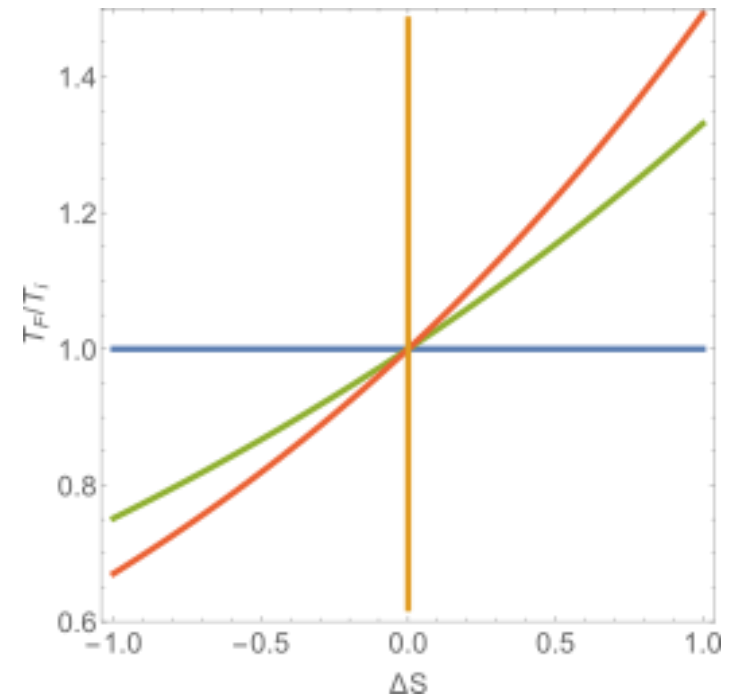
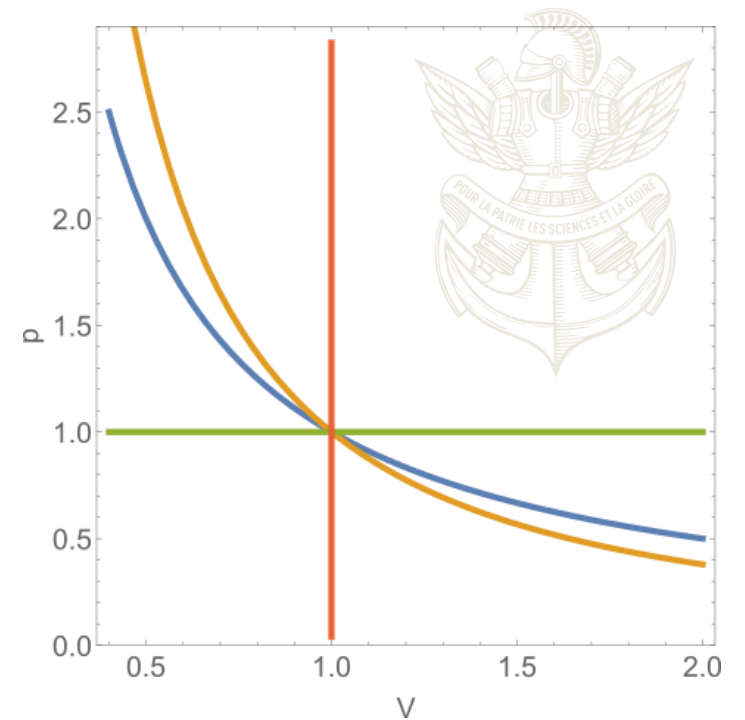
**Isochorus**

$$V = \text{cst}$$

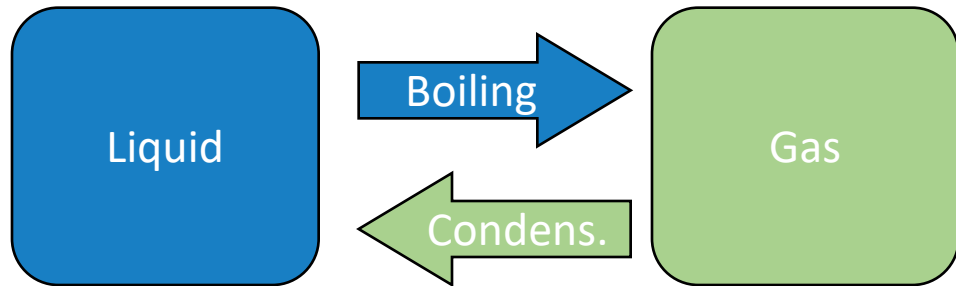


$$T_f = T_i \exp \Delta S / C_v$$

$$\Delta p = \frac{nR}{V} \Delta T$$



# Phase change



Phase change happen when... ?

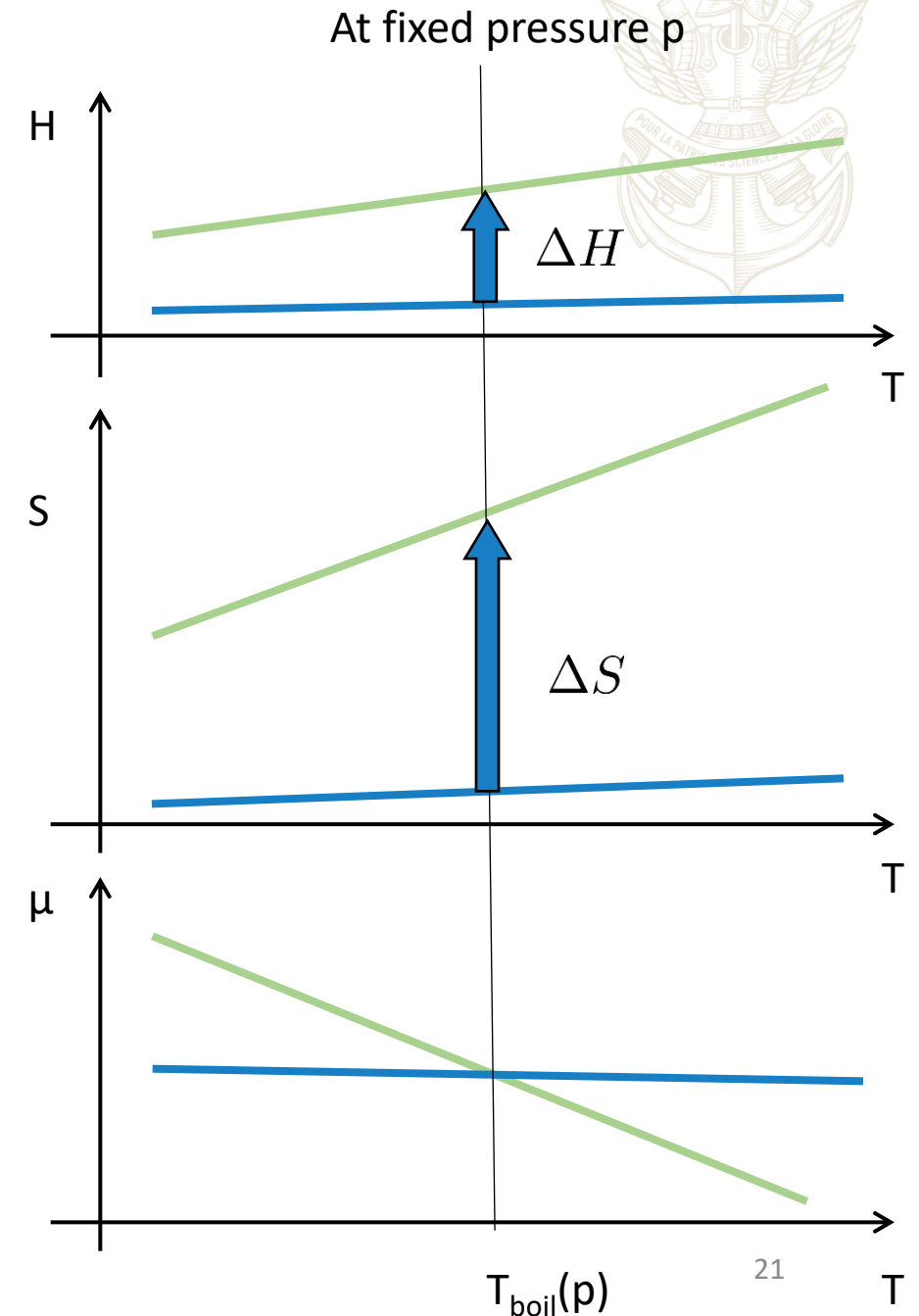
$$\mu_{\varphi_1}(T, p, \dots) = \mu_{\varphi_2}(T, p, \dots)$$

Pure body : change at fixed temperature

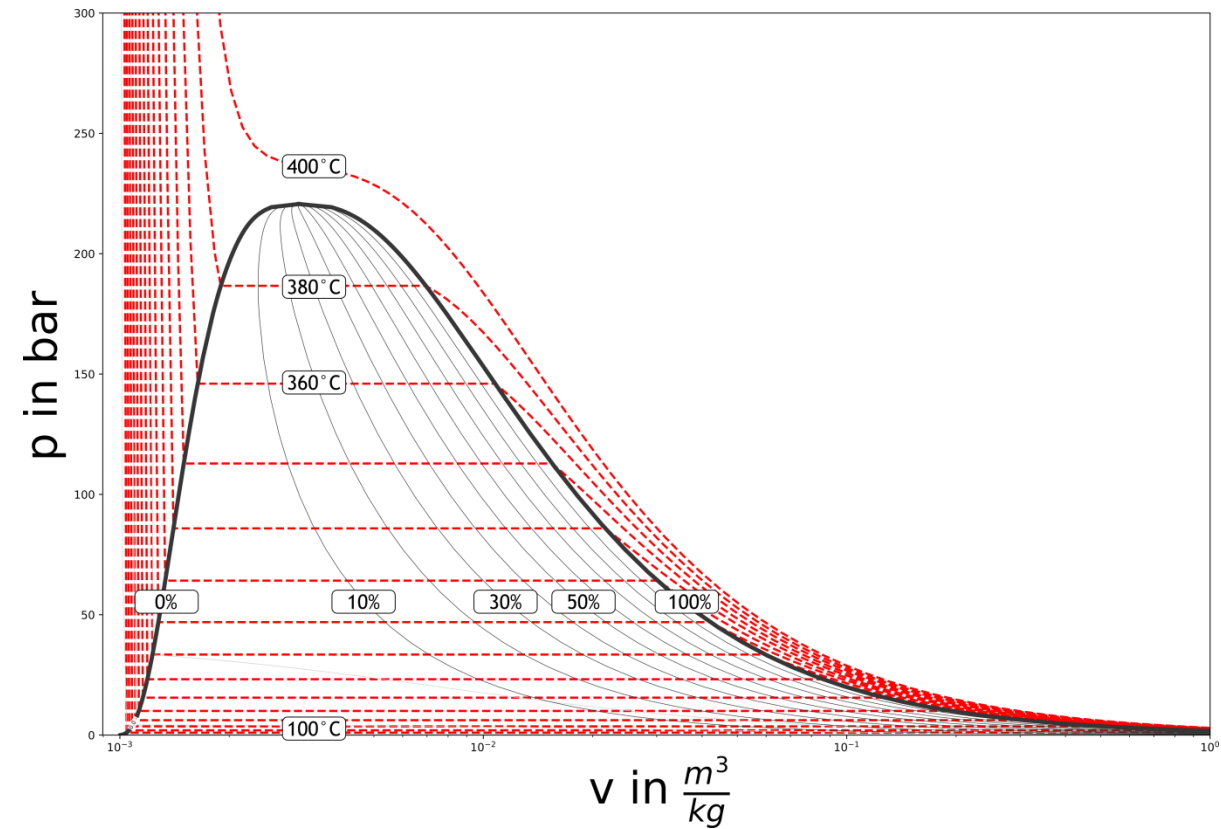
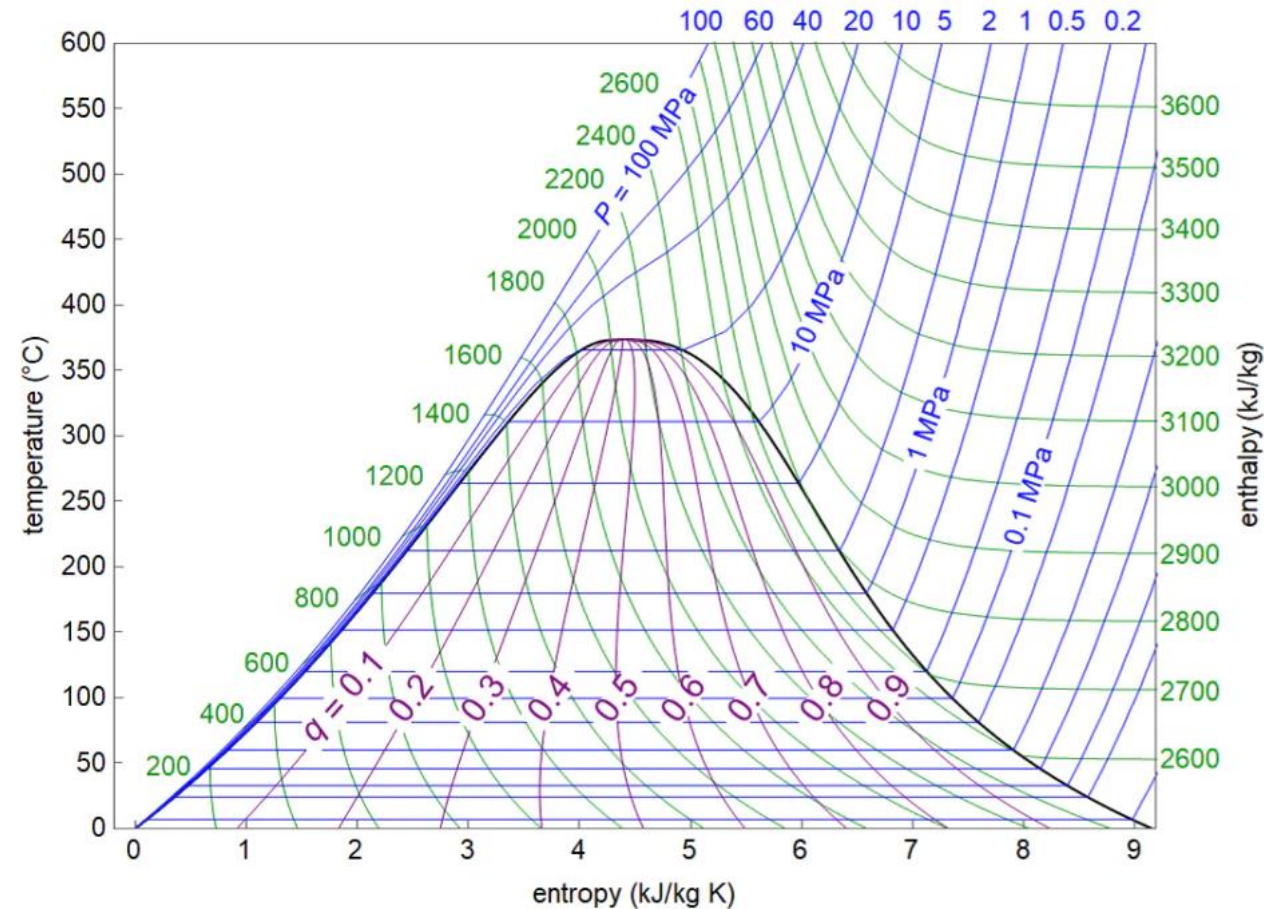
Amount of heat required for full change (*latent heat*)  $\Delta H$

Amount of entropy required for full change  $\Delta S$

$$\Delta S = \frac{\Delta H}{T} \quad \text{Reversible transformation}$$



# Phase change – diagram view



# Lecture 4

## Heat engines

I. What, why, where ?

II. Thermodynamics framework

III. Standard cycles and their applications

IV. Perspectives



# Air-standard assumptions

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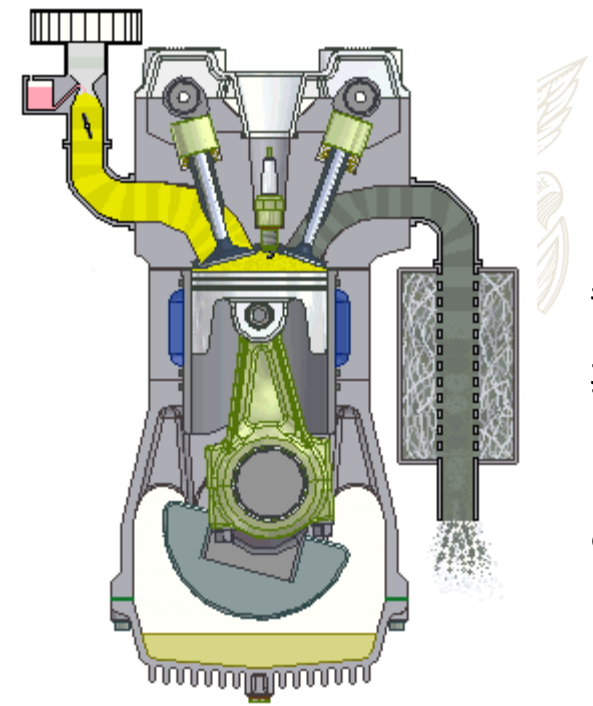
- Working fluid = pure air = ideal gas.
- Combustion = heat input from an external source
- Cycle = closed, with the same air remaining in the system. Intake and exhaust processes are not considered.
- All processes are reversible



# Otto cycle - theory

(aka Beau de Rochas cycle)

Internal combustion engine (ICE)  
with spark ignition (SI)



Source : wikipedia

(0-1: intake)

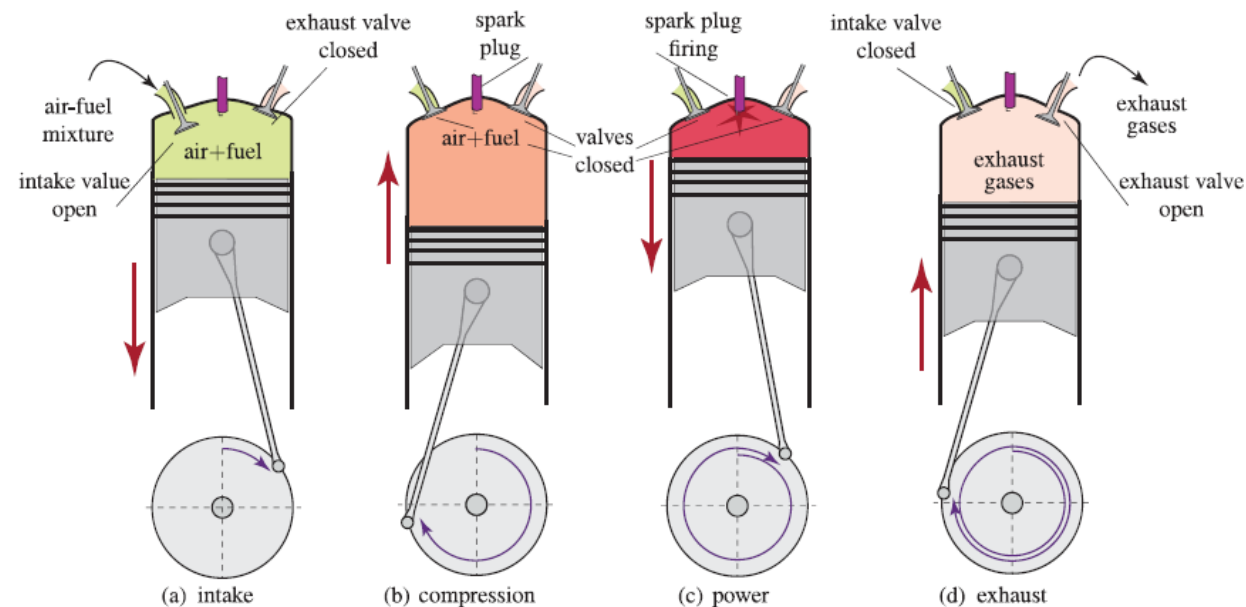
1-2: compression (isentropic)

2-3: ignition (**isochorous**) from a spark plug

3-4: expansion (isentropic)

4-1: ideal heat rejection (isochorous)

(1-0: exhaust)



# Otto cycle - theory

(0-1: intake)

1-2: compression (isentropic)

2-3: ignition (**isochorous**) from a spark plug

3-4: expansion (isentropic)

4-1: ideal heat rejection (isochorous)

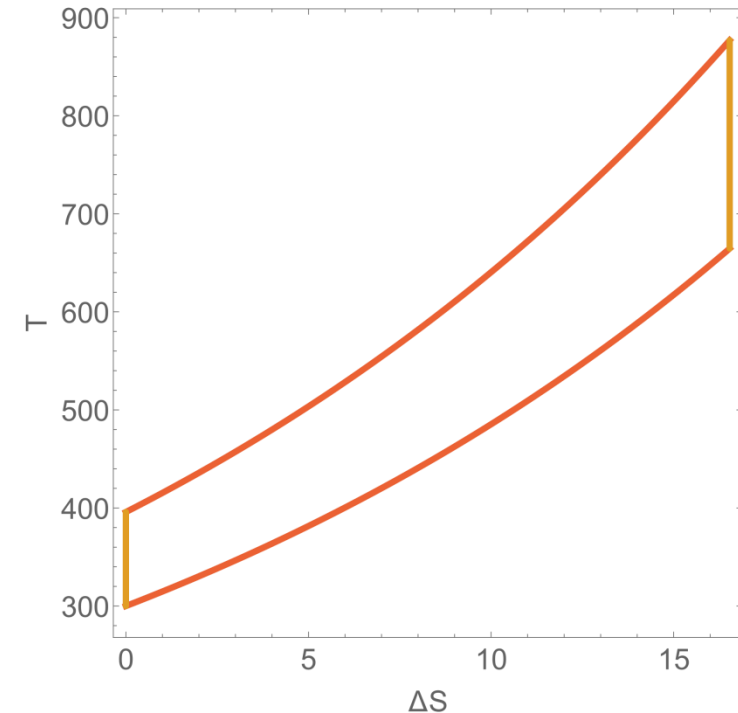
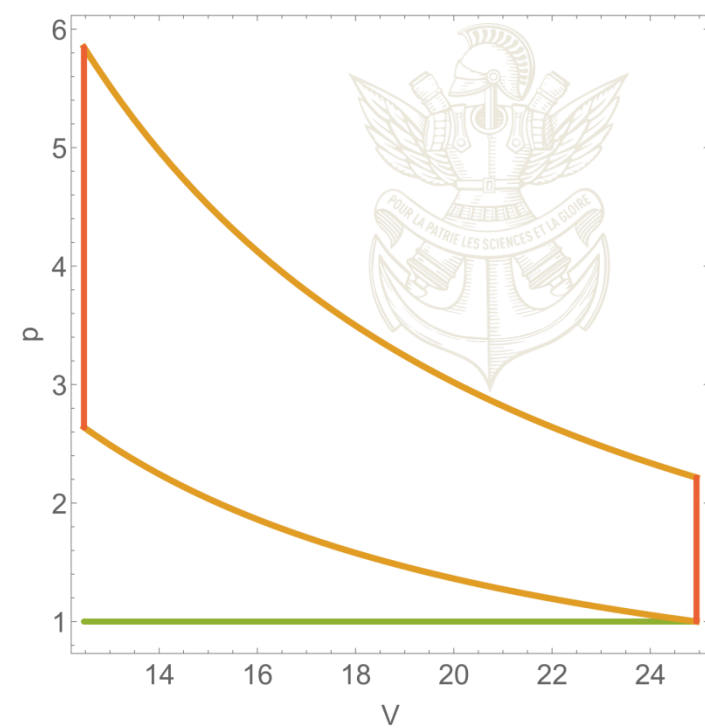
(1-0: exhaust)

(Volumic) compression ratio

$$r = \frac{V_1}{V_2}$$

Efficiency

$$\eta = 1 - \frac{1}{r^{\gamma-1}}$$



# 2 stroke versus 4 stroke

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Same thermodynamic cycle, different implementation

4 stroke : 1 cycle = 2 full rotations

Stroke = piston move

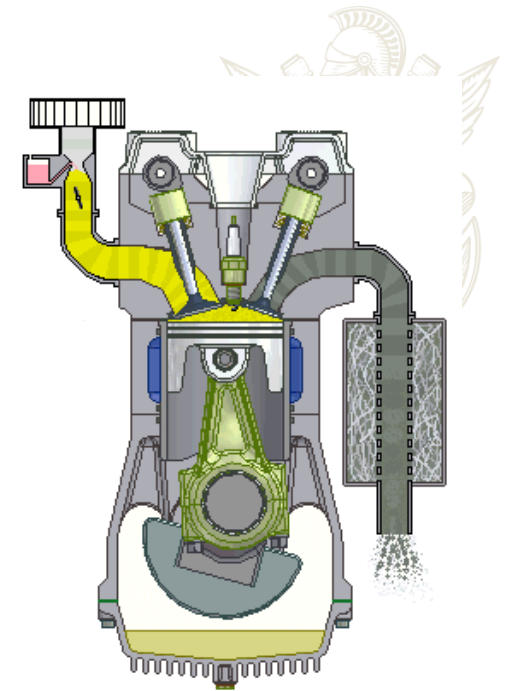
2 stroke : 1 cycle = 1 full rotation

Intake & exhaust at the same time

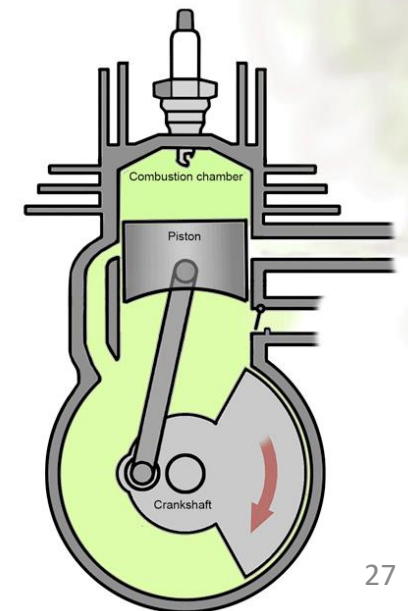
Lighter, less complex

Hand-carried applications (chainsaw...)

Lubricating oil burnt with fuel (« oil mix »)



Source : wikimedia



Source: UN Environment Program.

# Otto cycle - reality

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## Idealized vs real cycle

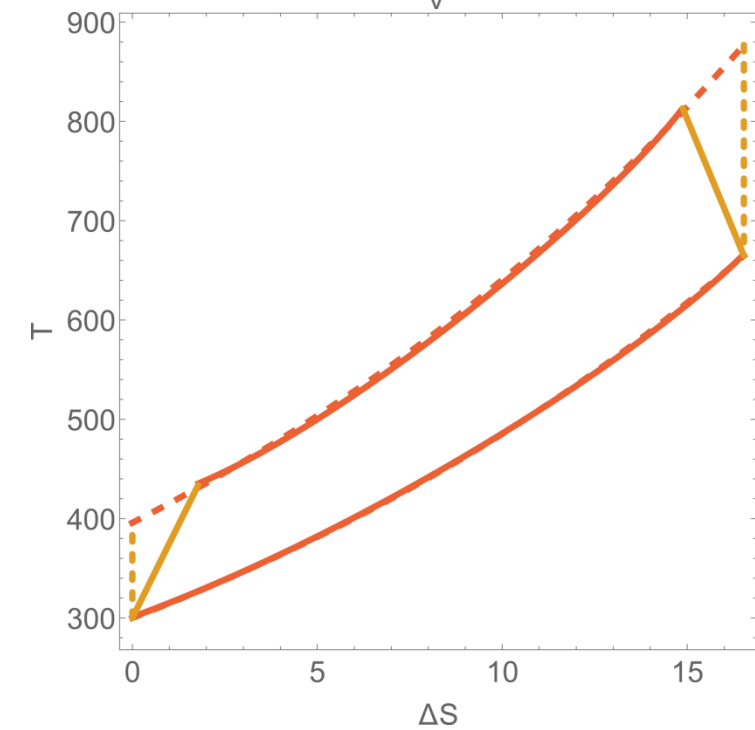
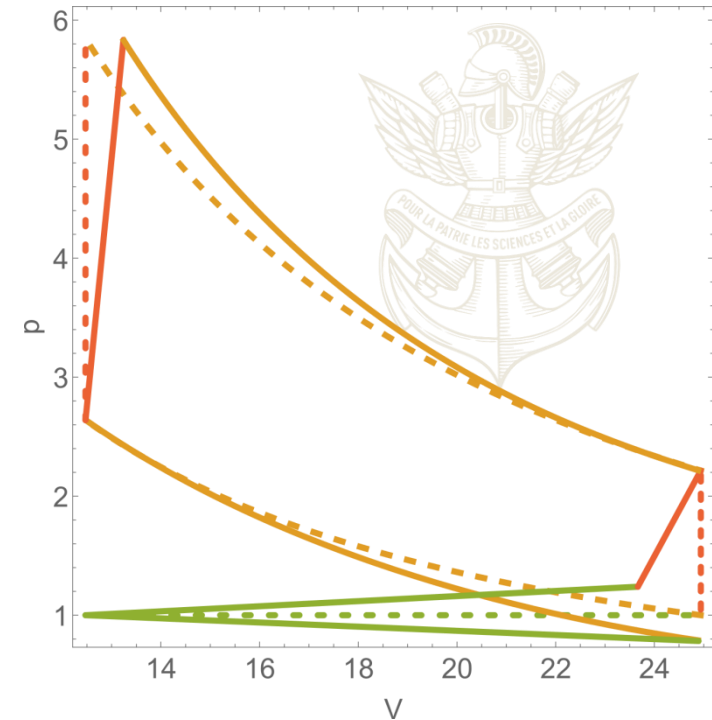
compressions are not fully isentropic  
heat rejection & exhaust at the same time

Mostly used in light vehicles

Compression ratio  $r \sim 10$ , limited by self-ignition (“knock”)

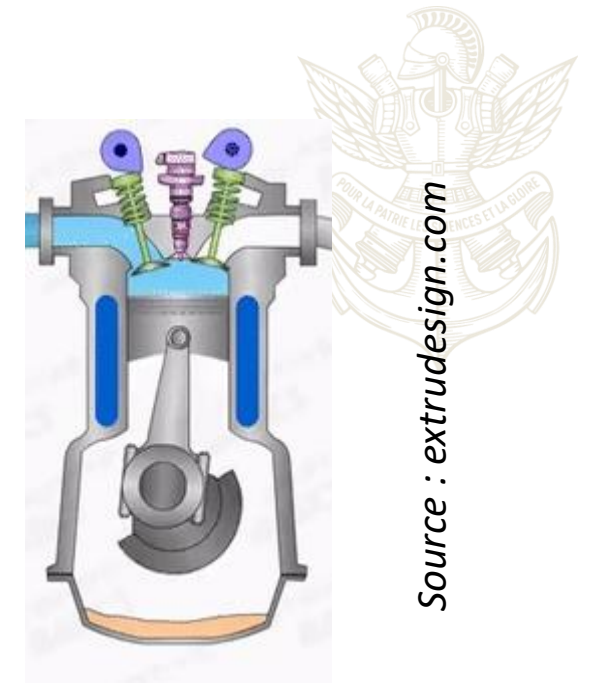
A turbo charger can extract work from the exhaust stream to pressurize intake air  
 $\Rightarrow$  increased yield

An intercooler can be used to reduce the intake air temperature and thus increase air density  
 $\Rightarrow$  increased power



# Diesel cycle - theory

Internal combustion engine (ICE)  
with compression ignition (CI)



(0-1: intake)

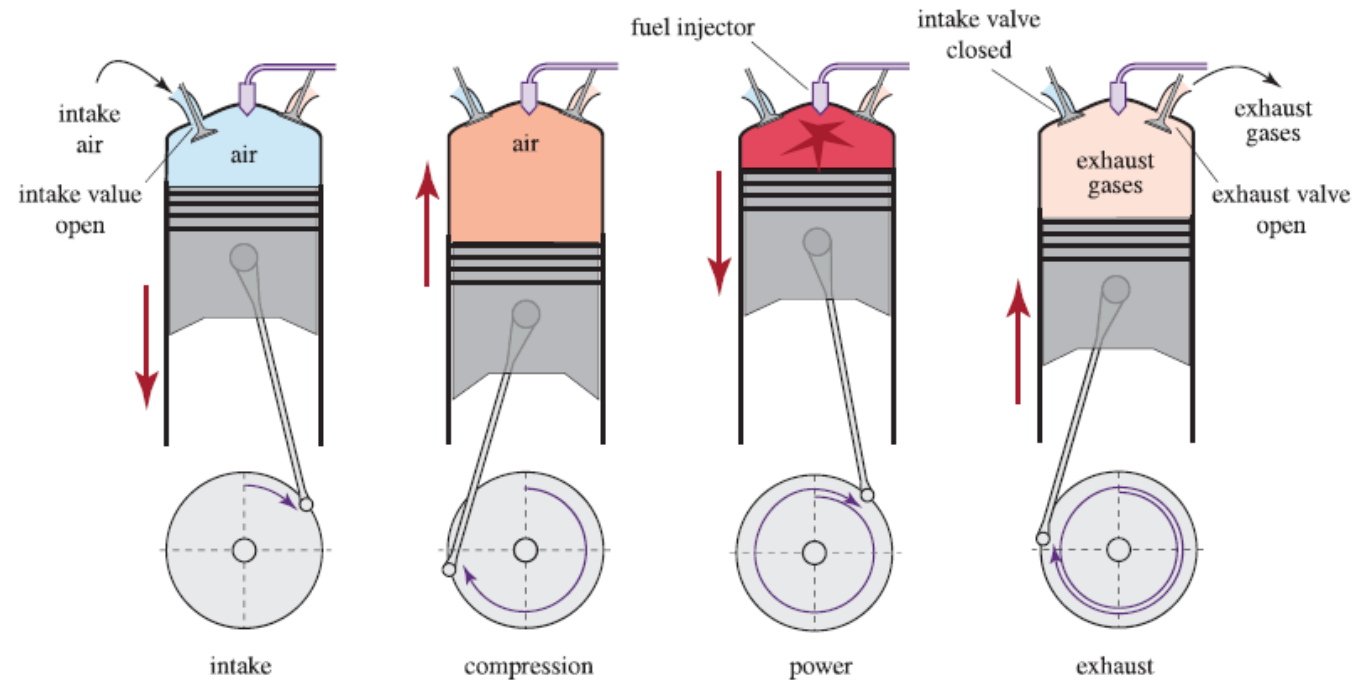
1-2: compression (isentropic)

2-3: ignition (**isobaric**)

3-4: expansion (isentropic) until  $V_1$

4-1: heat rejection (isochorous)

(1-0: exhaust)



# Diesel cycle - theory

(0-1: intake)

1-2: compression (isentropic)

2-3: ignition (**isobaric**)

3-4: expansion (isentropic) until  $V_1$

4-1: heat rejection (isochorous)

(1-0: exhaust)

(Volumic)  
compression ratio

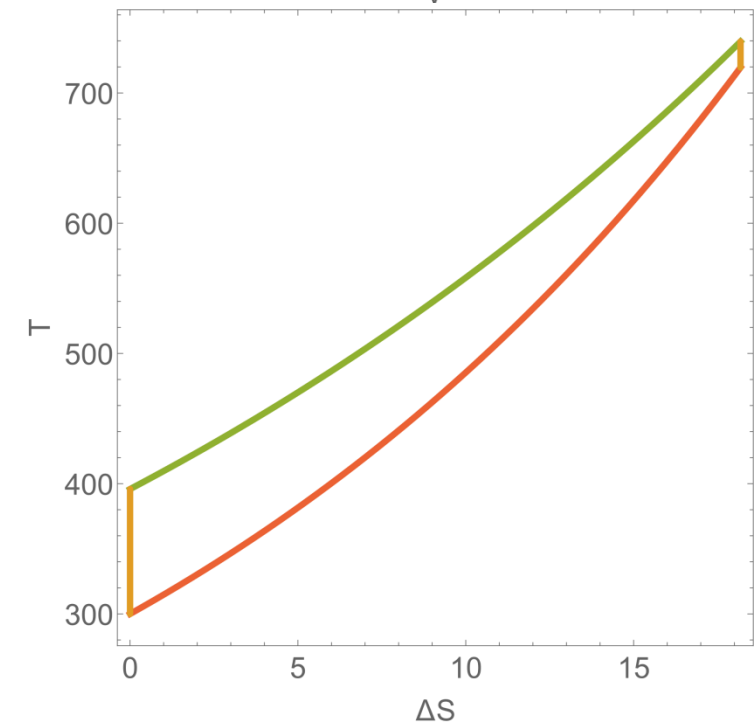
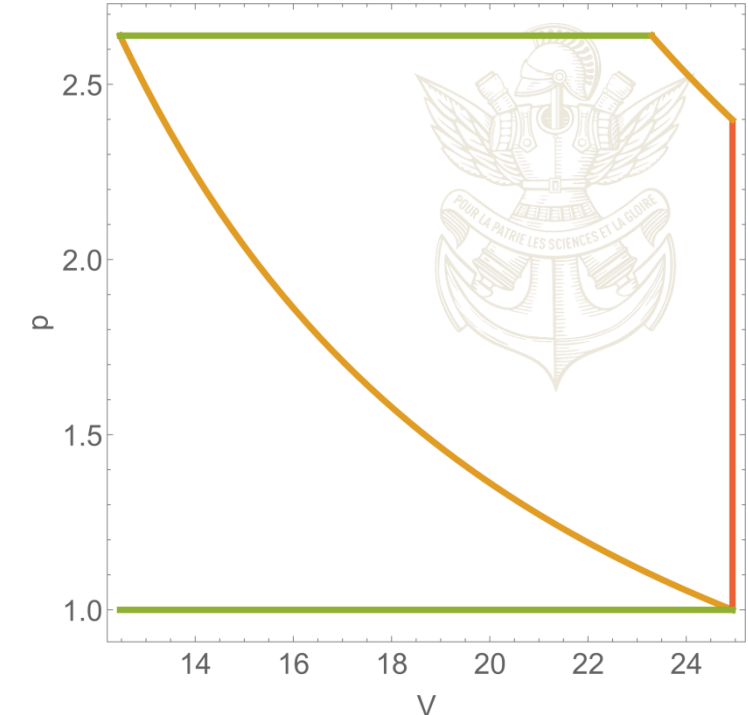
$$r = \frac{V_1}{V_2}$$

Cut off ratio

$$\beta = \frac{V_3}{V_2}$$

Efficiency

$$\eta = 1 - \frac{1}{r^{\gamma-1}} \frac{\beta^{\gamma} - 1}{\gamma(\beta - 1)}$$



# Diesel cycle - application

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Higher compression ratio (between 15 & 23)

→ Higher efficiency

Higher torque

Less sensitive to fuel

## **Wärtsilä-Sulzer 14RT-flex96C**

2 stroke, 14 cylinders

2 300 t, 13.5m high, 27.3m large

80 080 kW @ 102 rpm

3.8 L/s

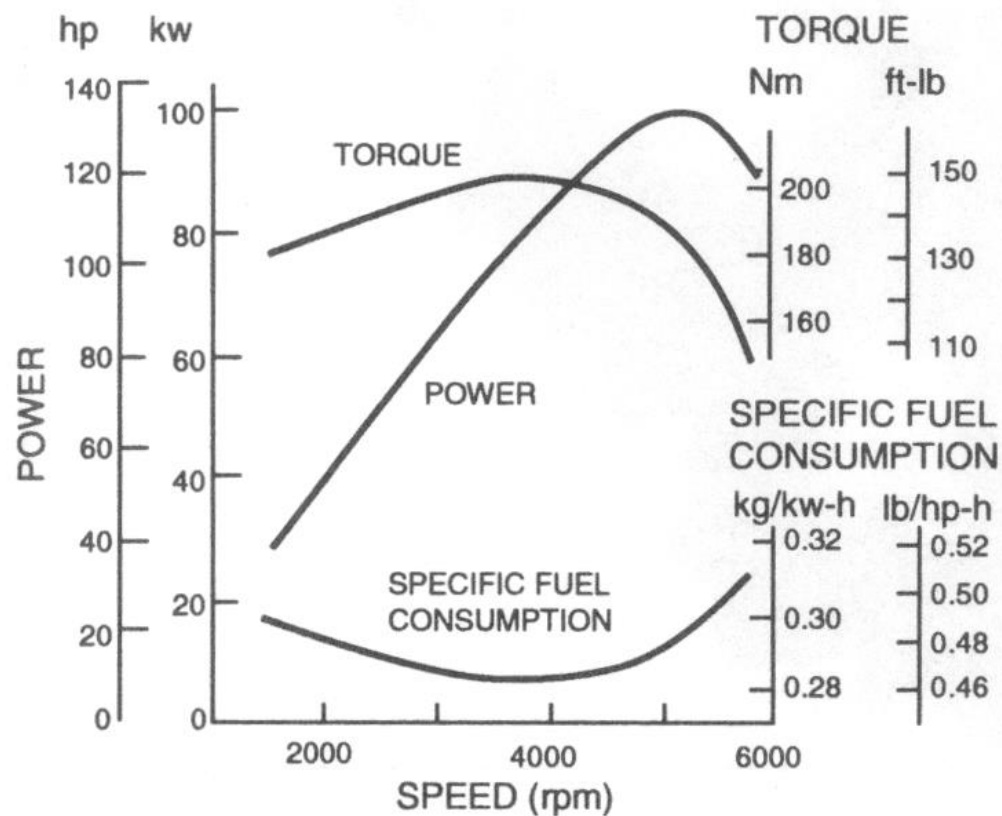
~ 50% efficiency



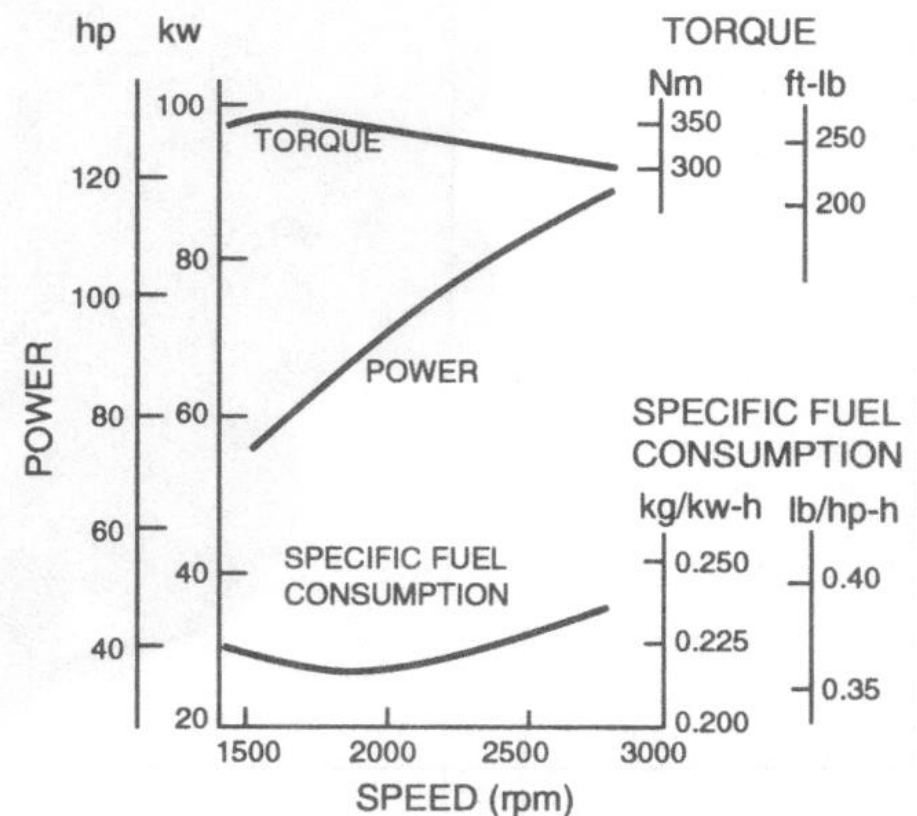


# Power & torque

$$P = \eta \times Q_{\text{comb}} \times \frac{1}{T_{\text{cycle}}} = \Gamma \times \omega$$



Gasoline



Diesel



# Brayton / Joule cycle - theory



(intake)

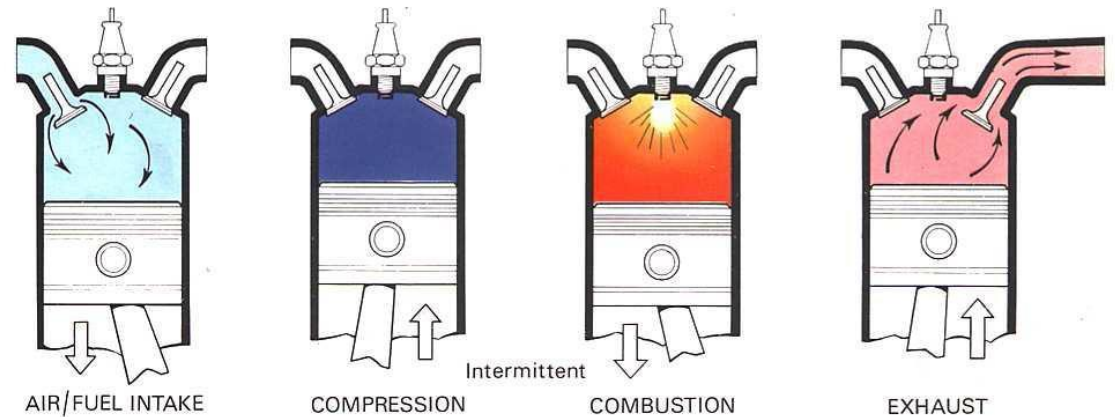
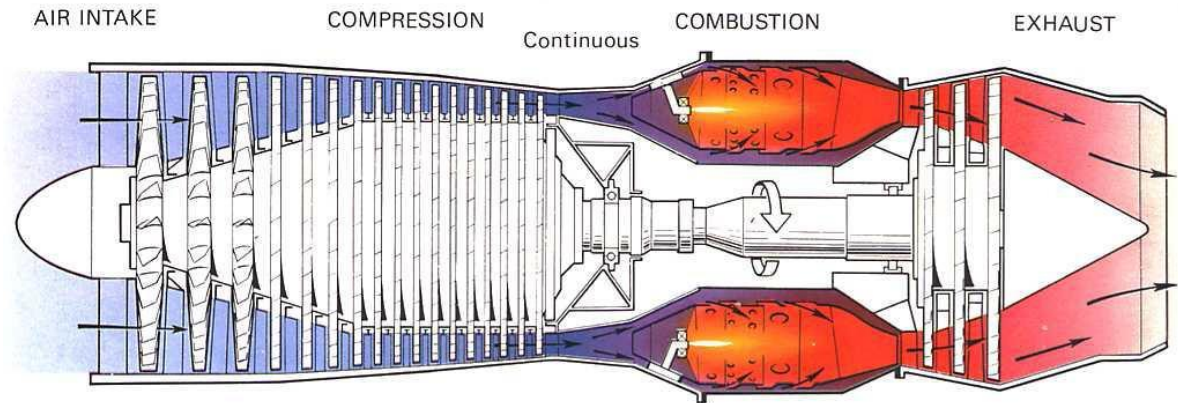
1-2: compression (isentropic)

2-3: ignition (**isobaric**)

3-4: expansion (isentropic)

4-1: ideal heat rejection (**isobaric**)

(exhaust)



# Brayton / Joule cycle - theory

intake

1-2: compression (isentropic)

2-3: ignition (**isobaric**)

3-4: expansion (isentropic)

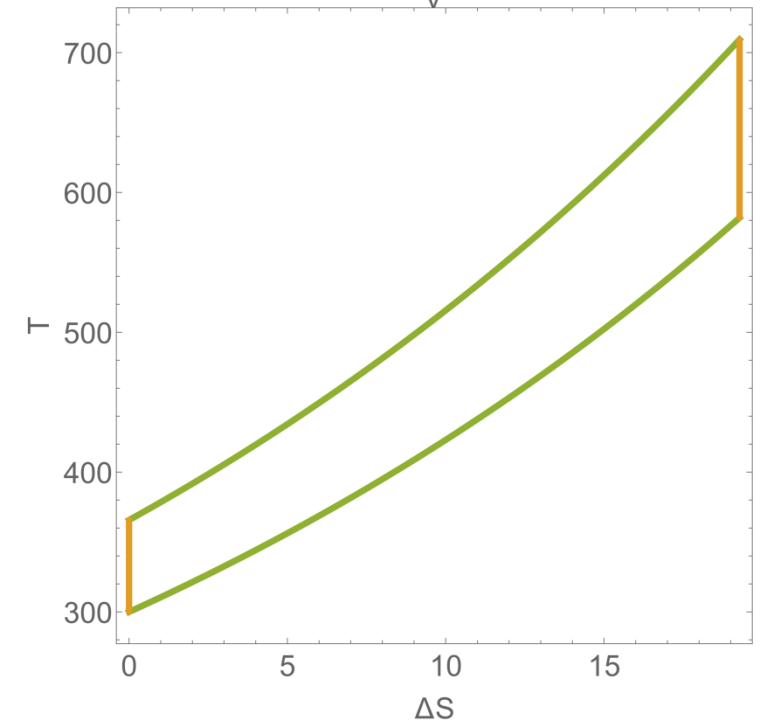
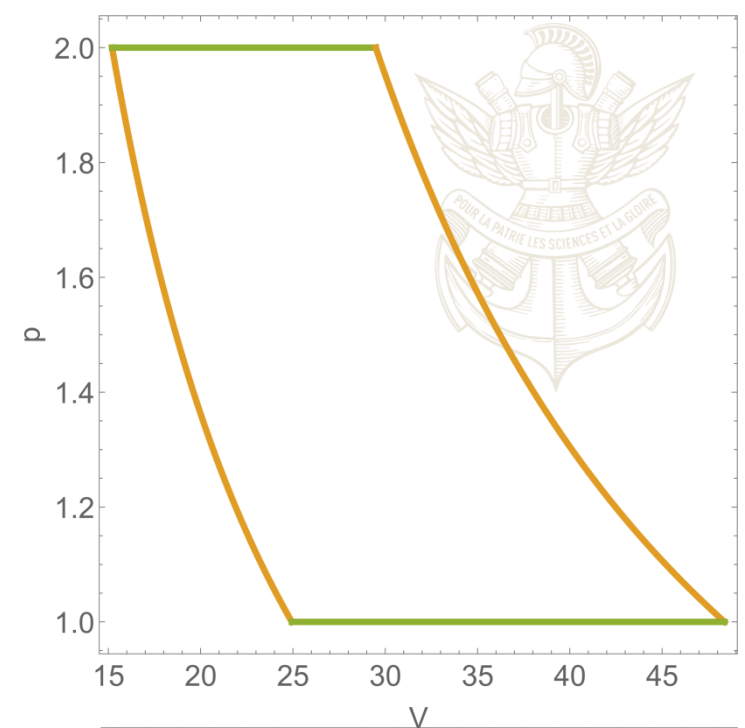
4-1: ideal heat rejection (**isobaric**)

exhaust

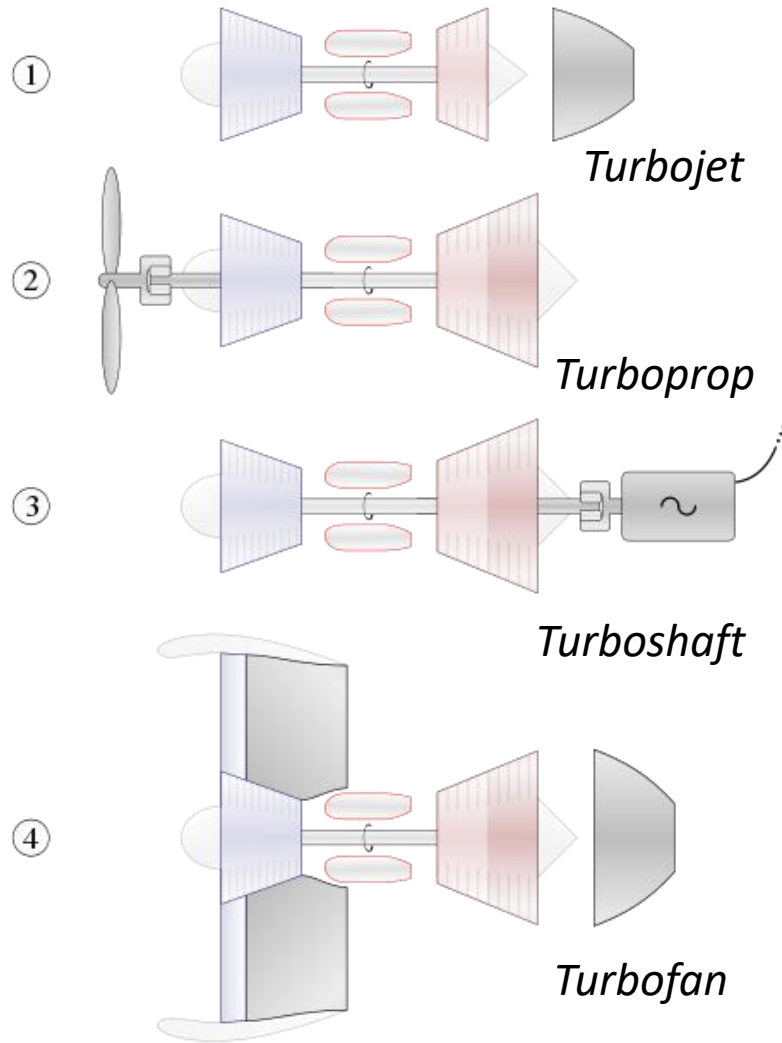
(Pressure) compression ratio

$$r_p = \frac{p_2}{p_1}$$

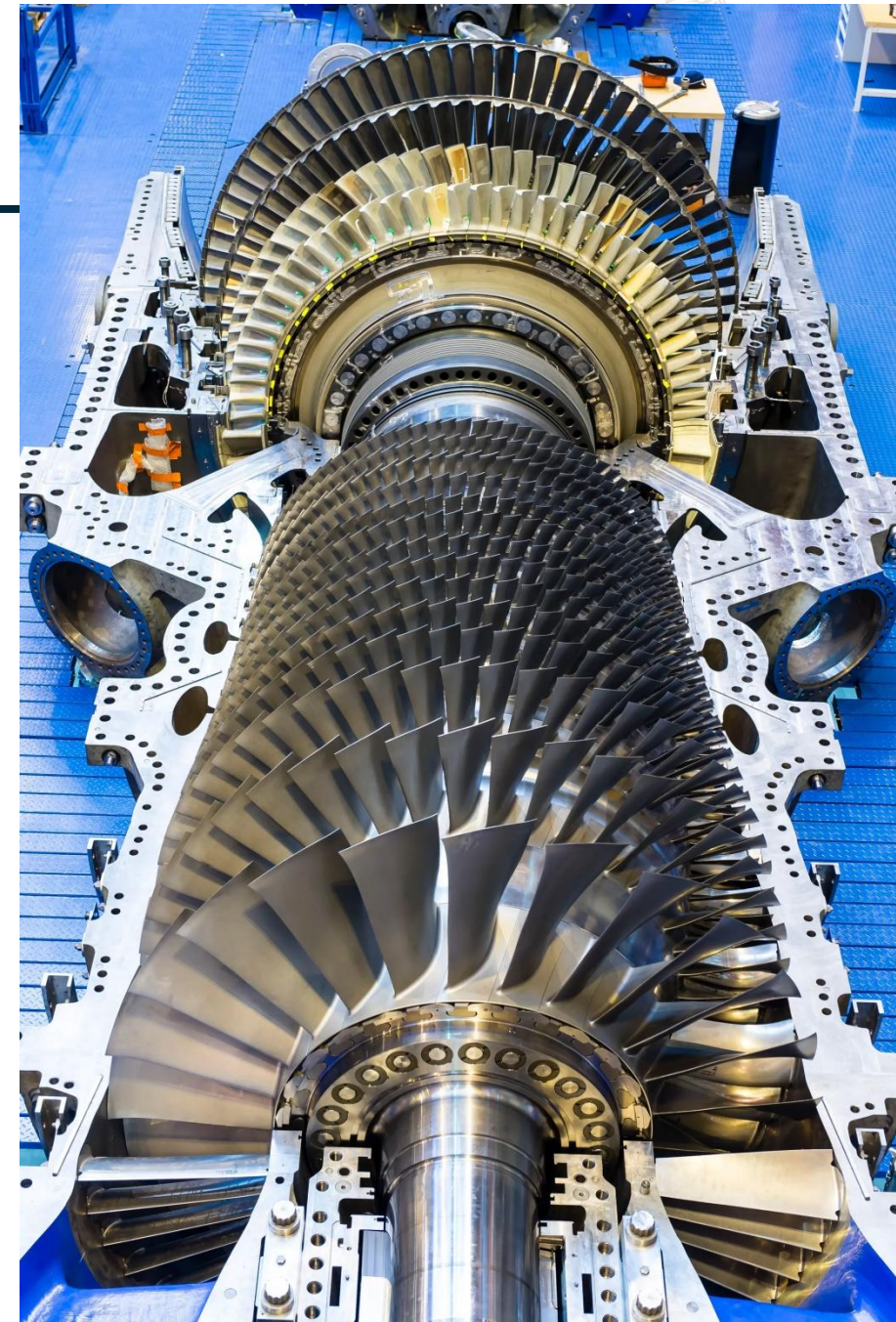
$$\eta = 1 - \frac{1}{r_p^{\frac{\gamma-1}{\gamma}}}$$



# Brayton cycle - application



**GE Harriet 9HA**  
571 MW – 44% efficiency  
Ramp rate: 88 MW/min



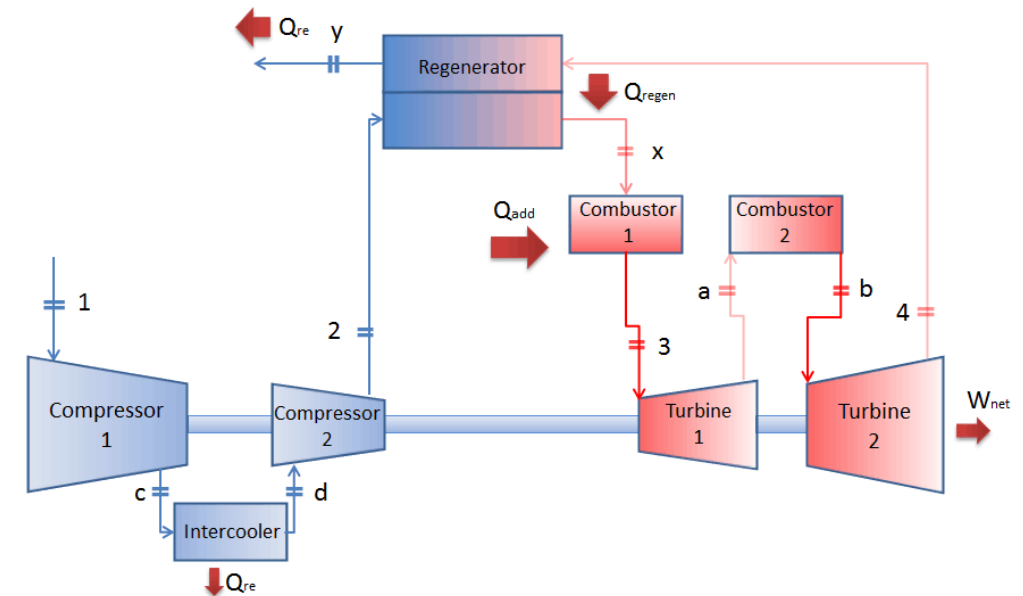
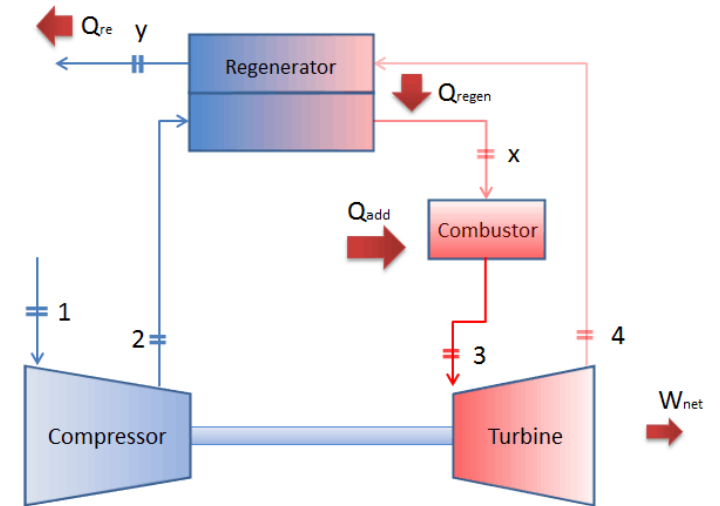
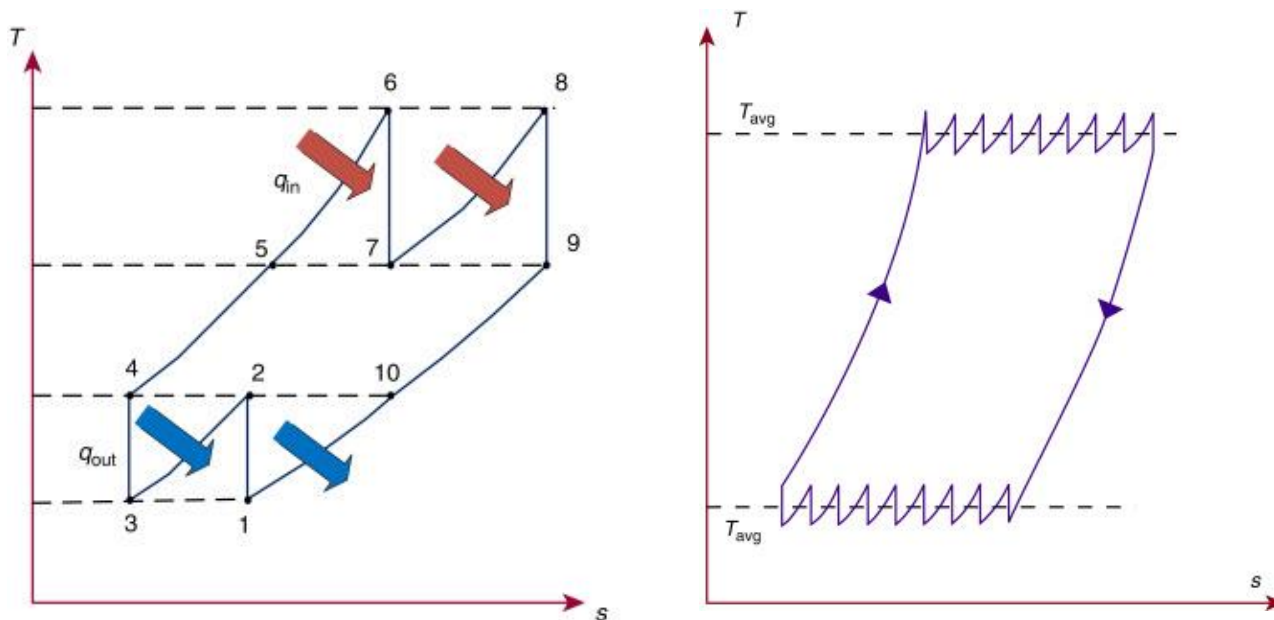


# Improving gas turbine efficiency

Avoid wasting exhaust heat

Reduce compression work :  
multi stage compressor with intercooler

Increase expansion work :  
multi stage combustion with reheaters



# Rankine cycle - theory

External combustion engine

1-2: **liquid** compression (isentropic)

2-3: heated and vaporized (isobaric)

3-4: expansion in a turbine (isentropic)

4-1: wet vapour condensation (isobaric)

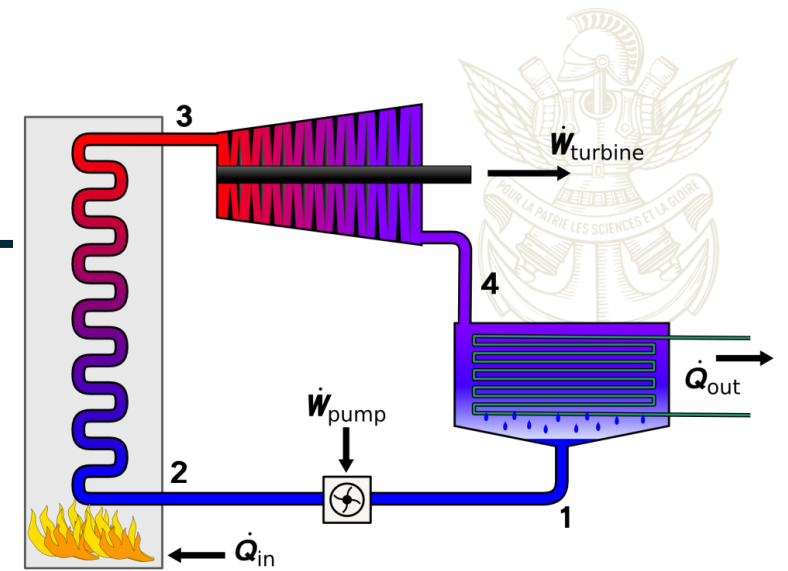
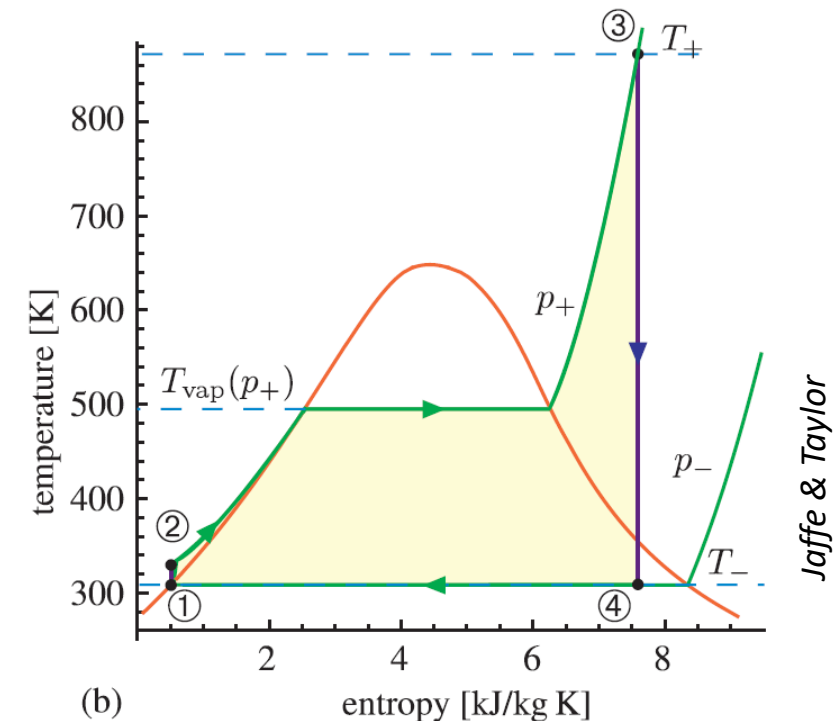
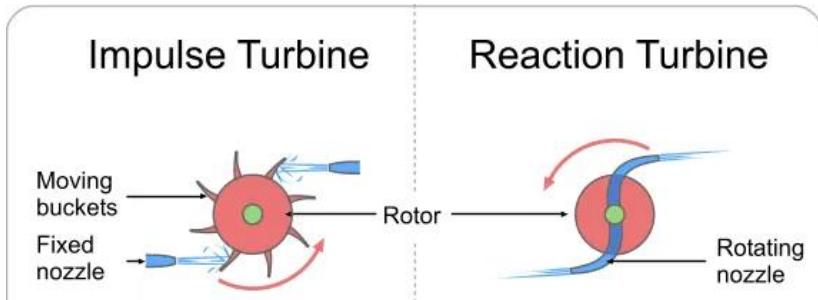


Figure : wikipedia





# Turbine design – impulse turbine



Driving force :  
velocity

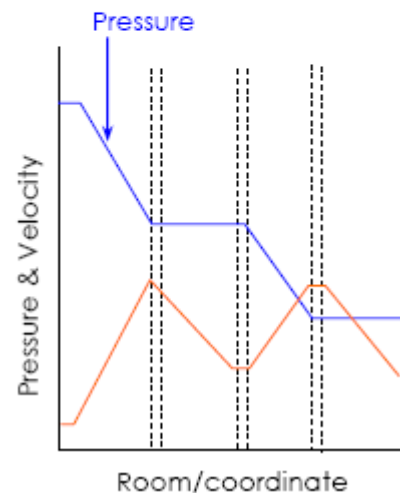
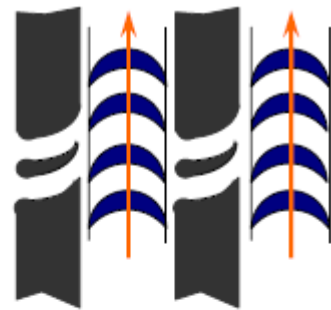
Fixed Nozzle  
converts  $p \rightarrow v$

Moving blades  
extract velocity  
at cst pressure

Fixed blades  
redirect velocity  
at cst pressure

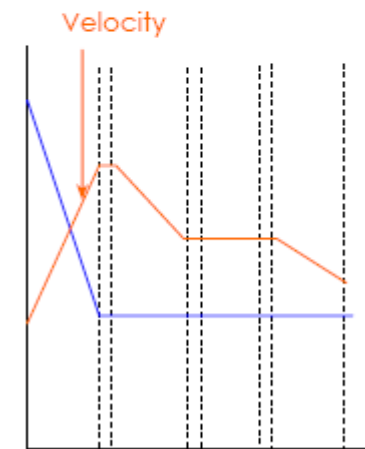
Rateau turbine

*Pressure compounding*



Curtis turbine

*Velocity compounding*

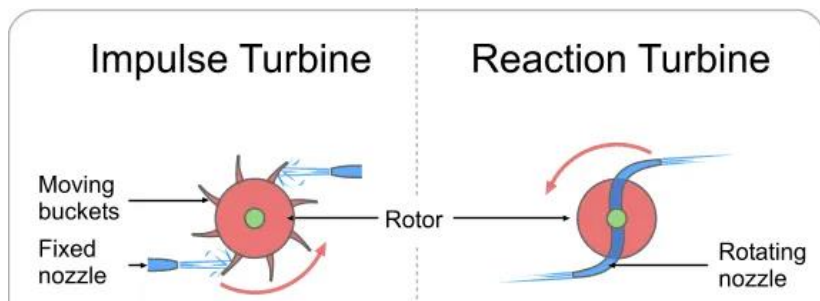


*Pressure – velocity compounding*





# Turbine design – reaction turbine



Driving force :  
velocity

Driving force :  
pressure

Fixed Nozzle  
converts  $p \rightarrow v$

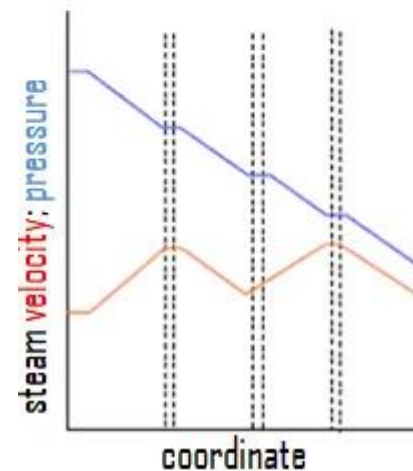
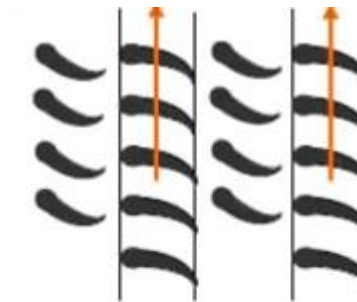
No “nozzle”

Moving blades  
extract velocity  
at cst pressure

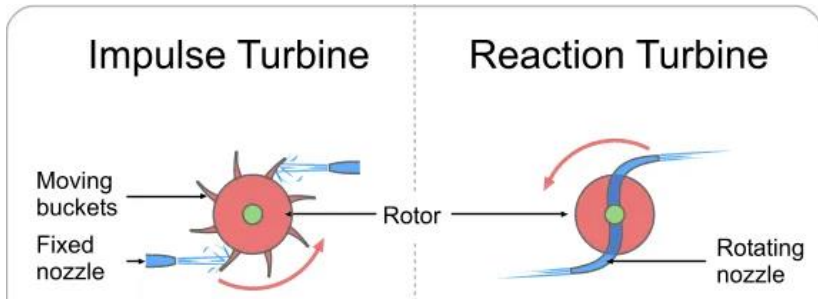
Moving blades  
due to pressure  
gradient

Fixed blades  
redirect velocity  
at cst pressure

Fixed blades  
due to pressure  
gradient



# Turbine design – in practice



Driving force :  
velocity

Fixed Nozzle  
converts  $p \rightarrow v$

Moving blades  
extract velocity  
at cst pressure

Fixed blades  
redirect velocity  
at cst pressure

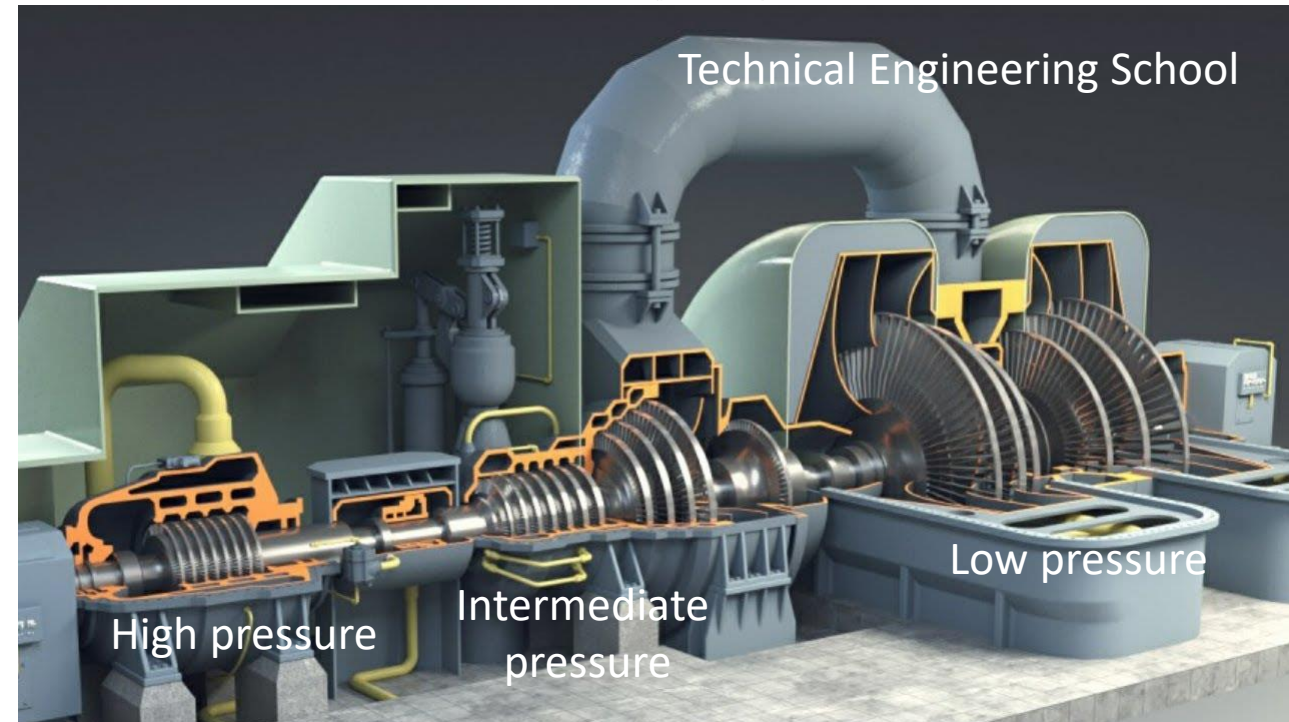
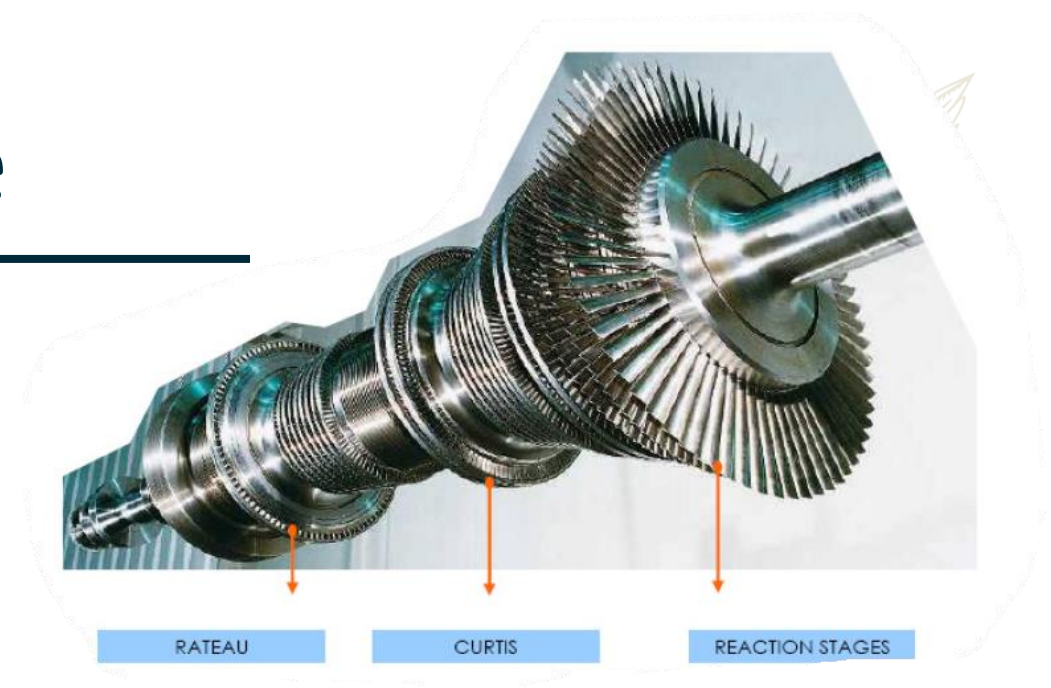
Reaction Turbine

Driving force :  
pressure

No “nozzle”

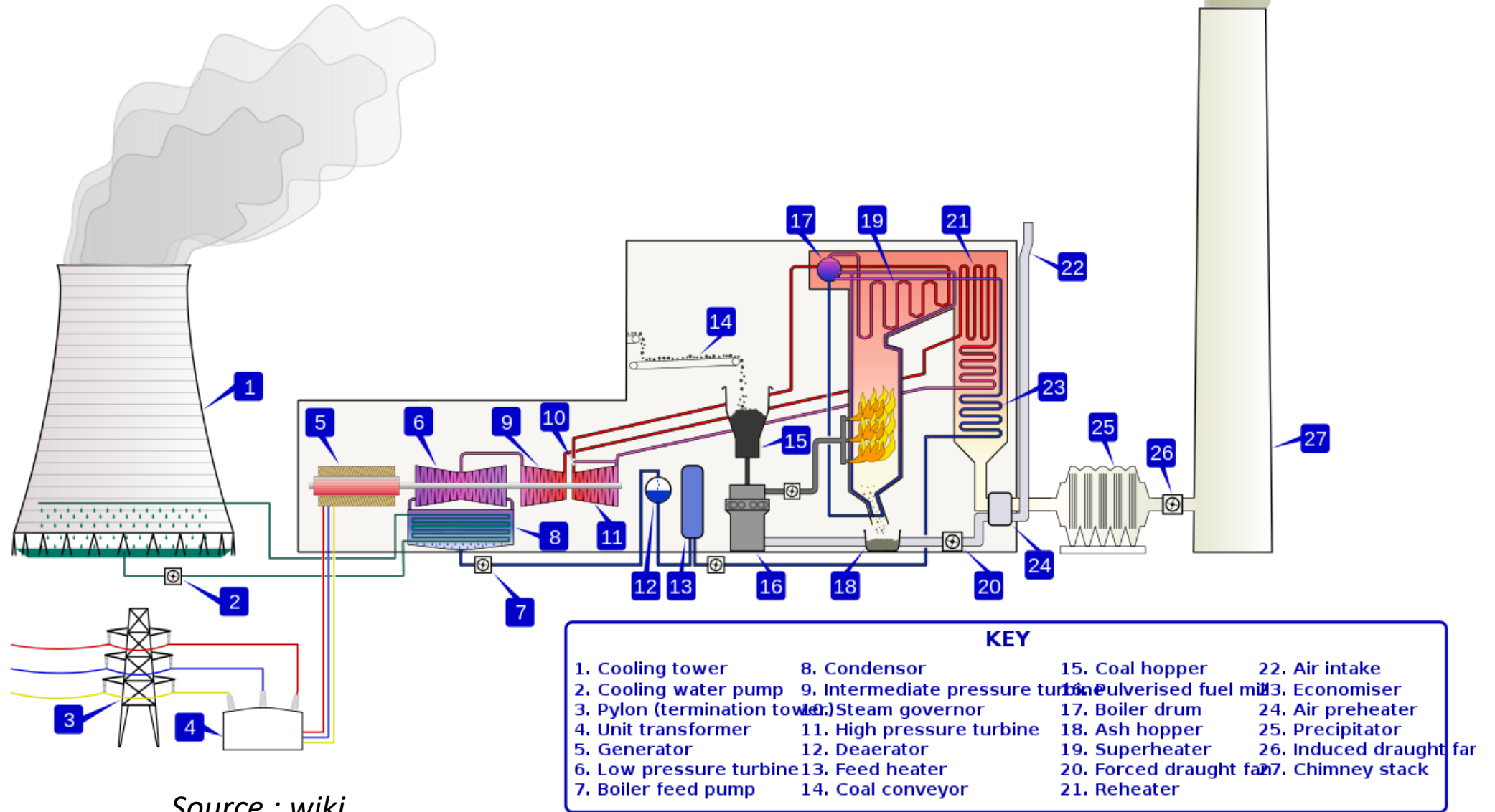
Moving blades  
due to pressure  
gradient

Fixed blades  
due to pressure  
gradient





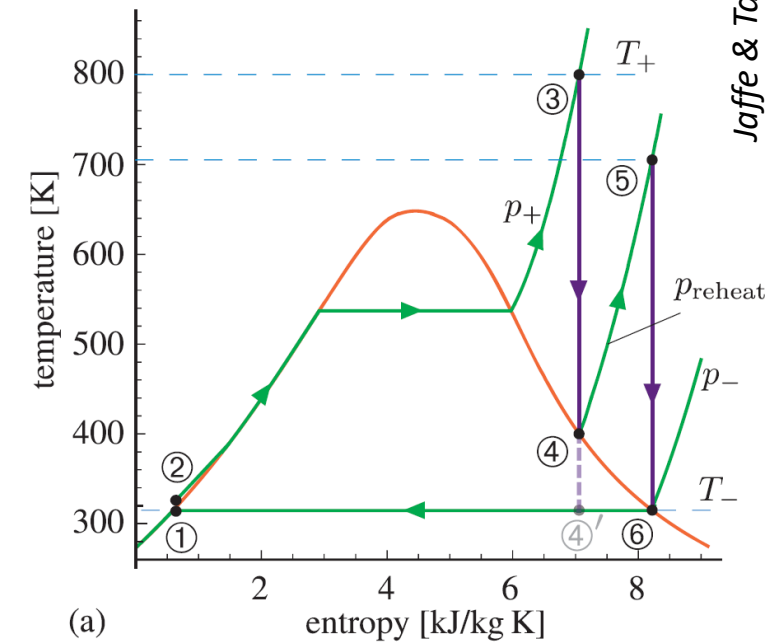
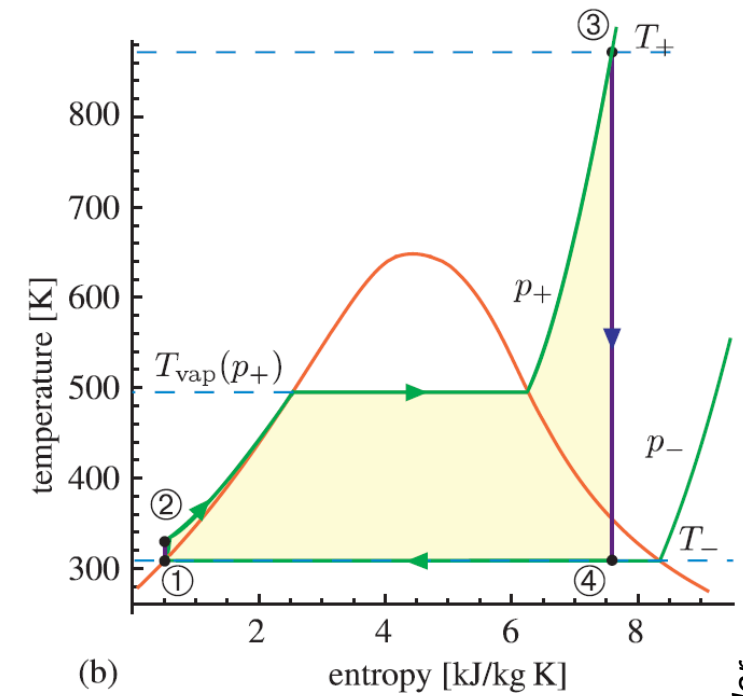
# Coal (and nuclear) power plant



Source : wiki

# Rankine cycle - improvements

Why is the last stage of the turbine most subject to eroding ?



# Rankine cycle - improvements



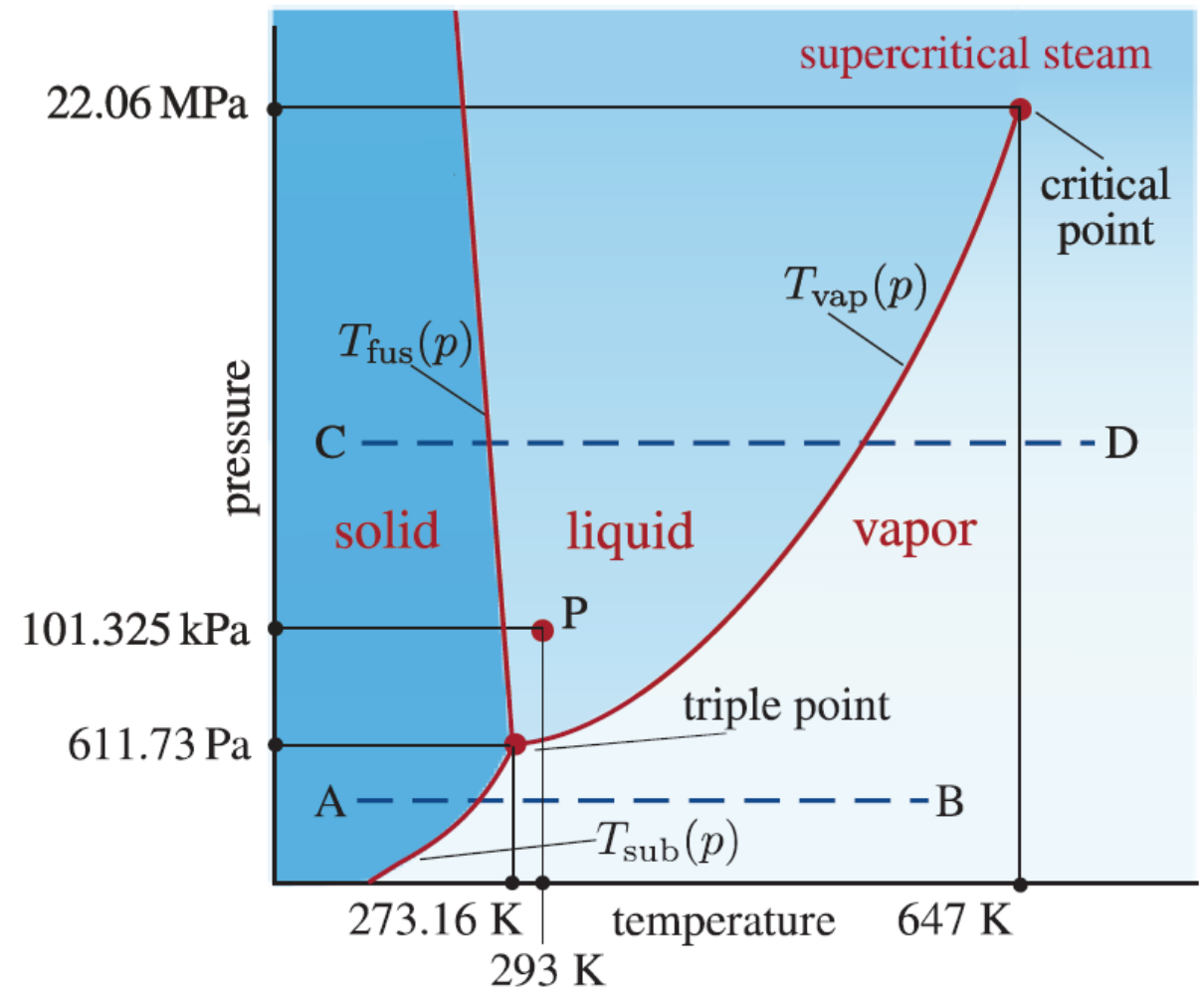
Above the critical point (221 bar, 374°C)  
no liquid – vapor phase change

Requires less enthalpy to reach high temperatures  
→ less heat input

Less entropy after the “boiler”  
→ more heat input

Efficiency : 42-45%

But strong material requirements !



# Rankine cycle - improvements

---



1957: first supercritical steam-electric generating unit in the world (Philo Power Plant, Ohio, USA)

2010 : Rheinhafen steam power plant (Karlsruhe, Germany)

47.5% net thermal efficiency  
919 MW of electricity (+220 MW heat)

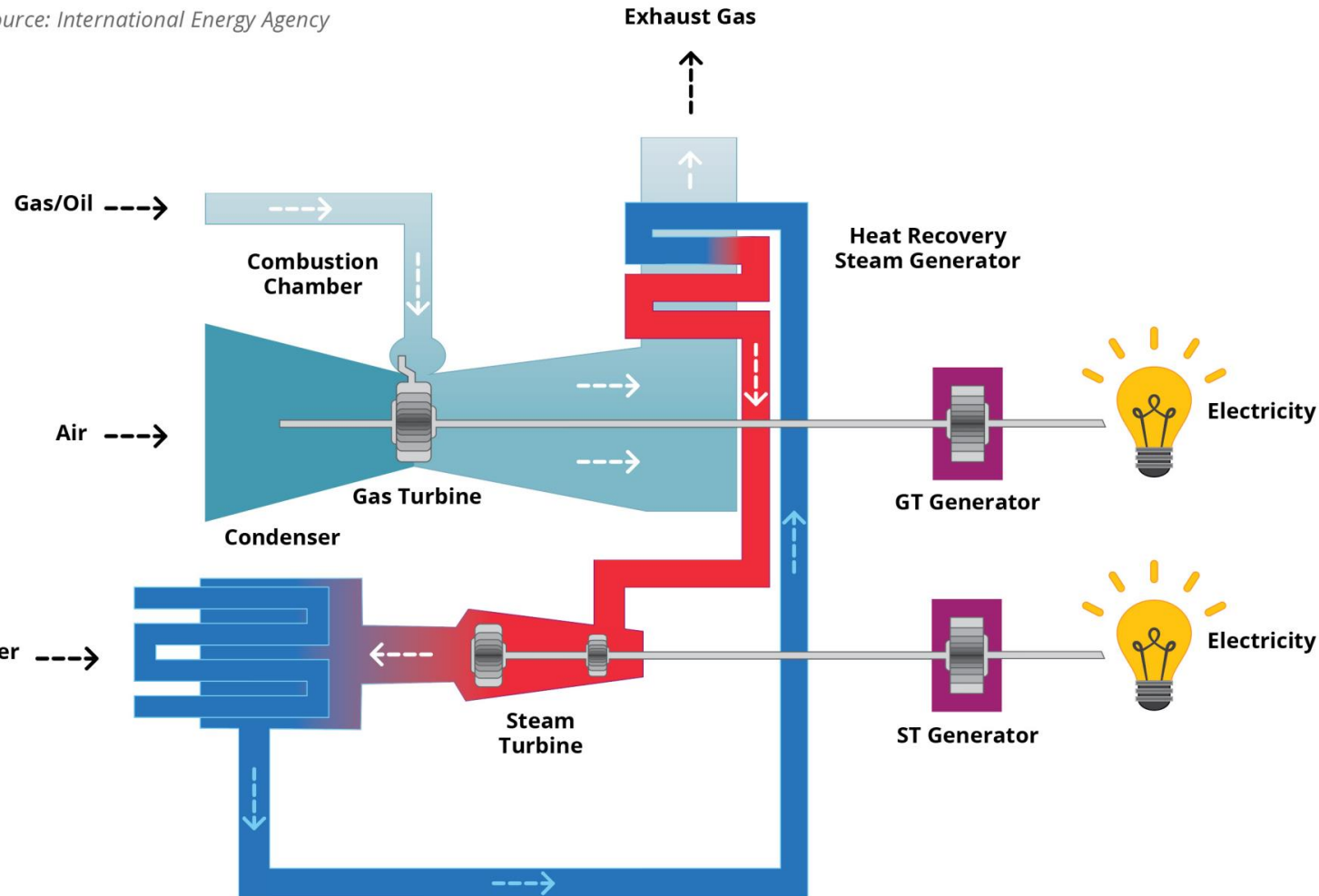
Steam > 600°C  
275 bar



# Combined cycle (CCGT)



Source: International Energy Agency



Gas turbine efficiency  $\eta_{GT}$

Steam turbine efficiency  $\eta_{ST}$

Heat exchange efficiency  $\beta$

$$\eta_{CCGT} = \frac{W_{GT} + W_{ST}}{Q_{\text{combustion}}} = \eta_{GT} + \beta(1 - \eta_{GT})\eta_{ST}$$

Up to 60%

# Combine cycle - application



## Gas turbine : Siemens SGT5-8000H (« H » class)

Gross Power Output	340 MW
Pressure ratio	19.2
Exhaust temperature	625 °C
Exhaust mass flow	820 kg/s
Net power output	530 MW

## Vapour turbine : Siemens SST-5000

Combined Cycle	120 MW to 500 MW
Conventional Steam	120 MW to 700 MW
Steam Temperature:	Up to 600° C
Pressure:	Up to 177 bar
Reheat Steam T :	Up to 600°

Gas and vapour on same rotation shaft

Net efficiency (in CC) 60%



# Lecture 4

## Heat engines

I. What, why, where ?

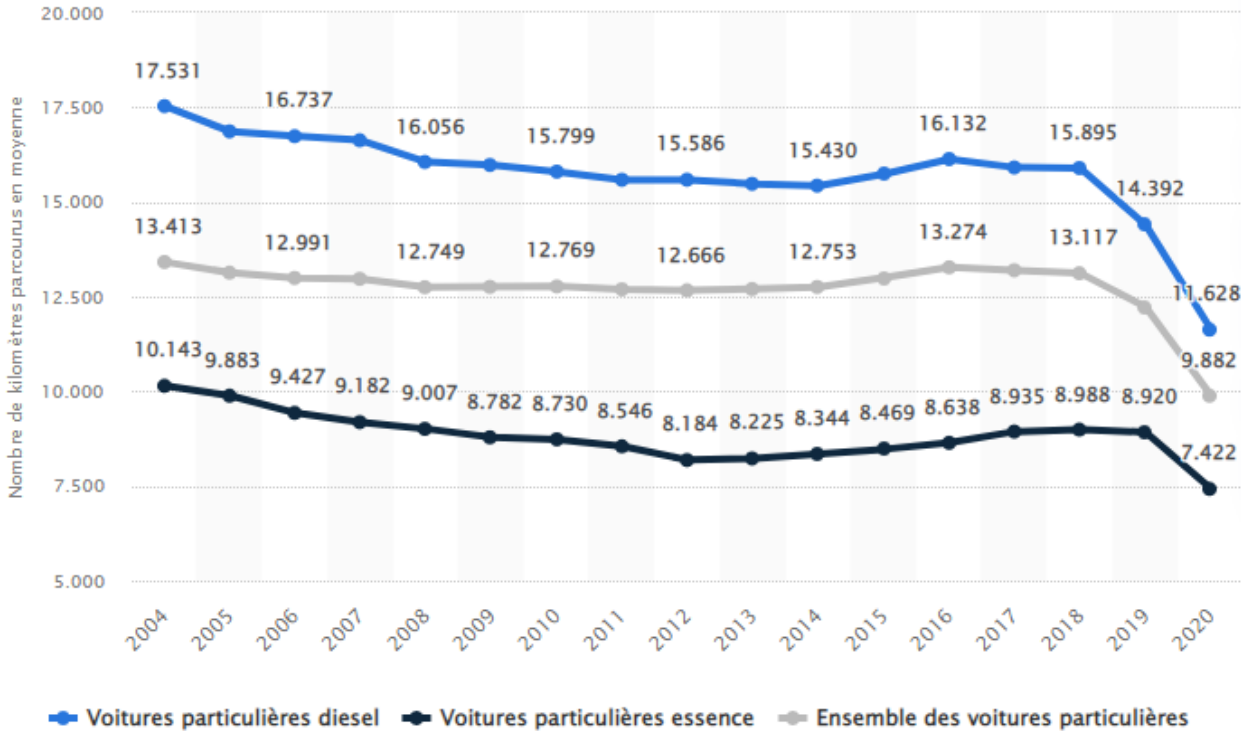
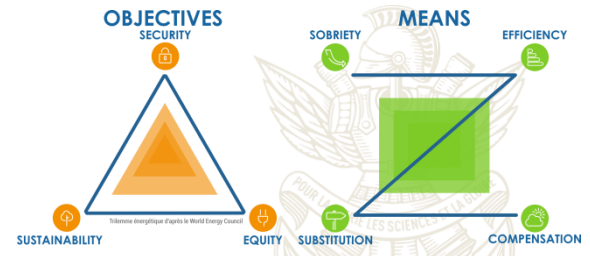
II. Thermodynamics framework

III. Standard cycles and their applications

IV. Perspectives

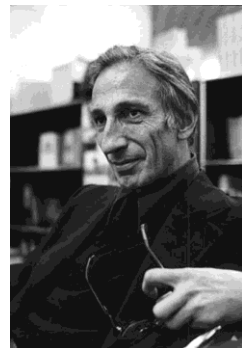


# Sufficiency - transport



13 000 km/year = 35 km/day

Infrastructures ?  
Daily planning ?

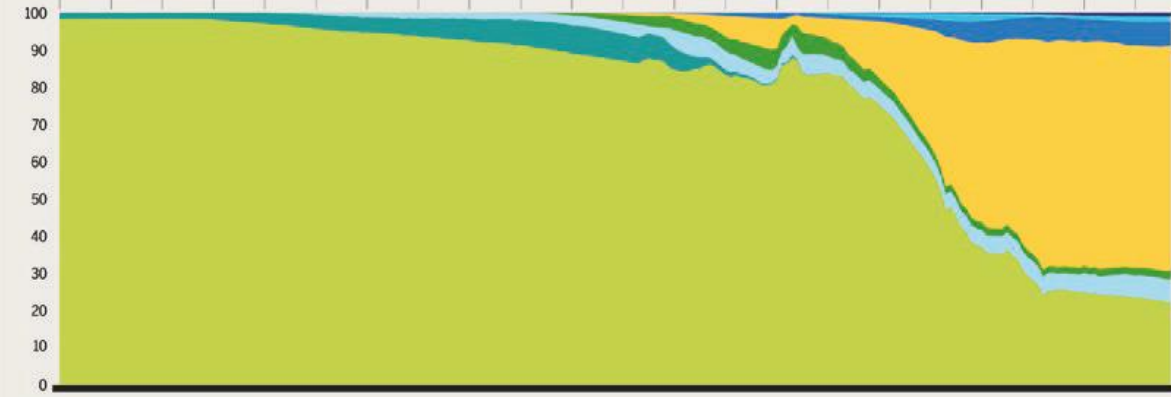
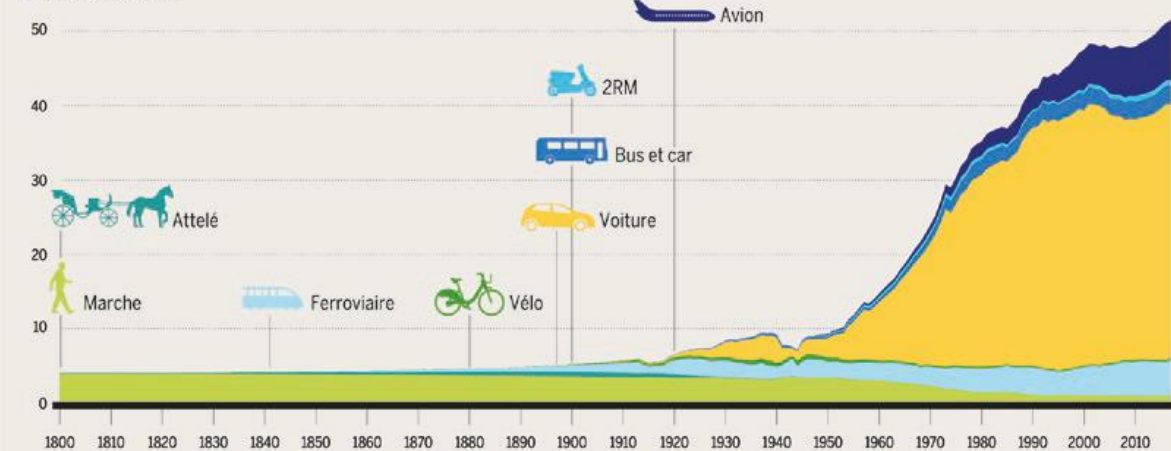


Ivan Illich, Energy & equity

## CHRONOLOGIE D'UNE ACCÉLÉRATION DE LA MOBILITÉ

Estimation du nombre de kilomètres parcourus par jour et par personne et part des modes de transport dans le temps de déplacement en France, de 1800 à 2017

### Km/jour/personne



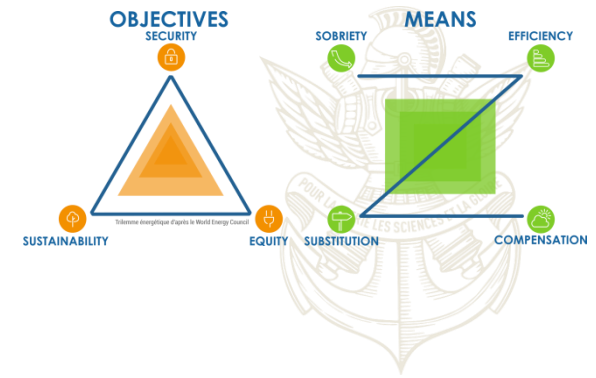
### Temps de déplacement (%)

2RM = deux-roues motorisés ; le terme voiture inclut également les véhicules utilitaires légers (VUL)

Source : Bigo, A., 2020. Les transports face au défi de la transition énergétique. Explorations entre passé et avenir, technologie et sobriété, accélération et ralentissement. Thèse. 340 pages.



# Sufficiency electricity

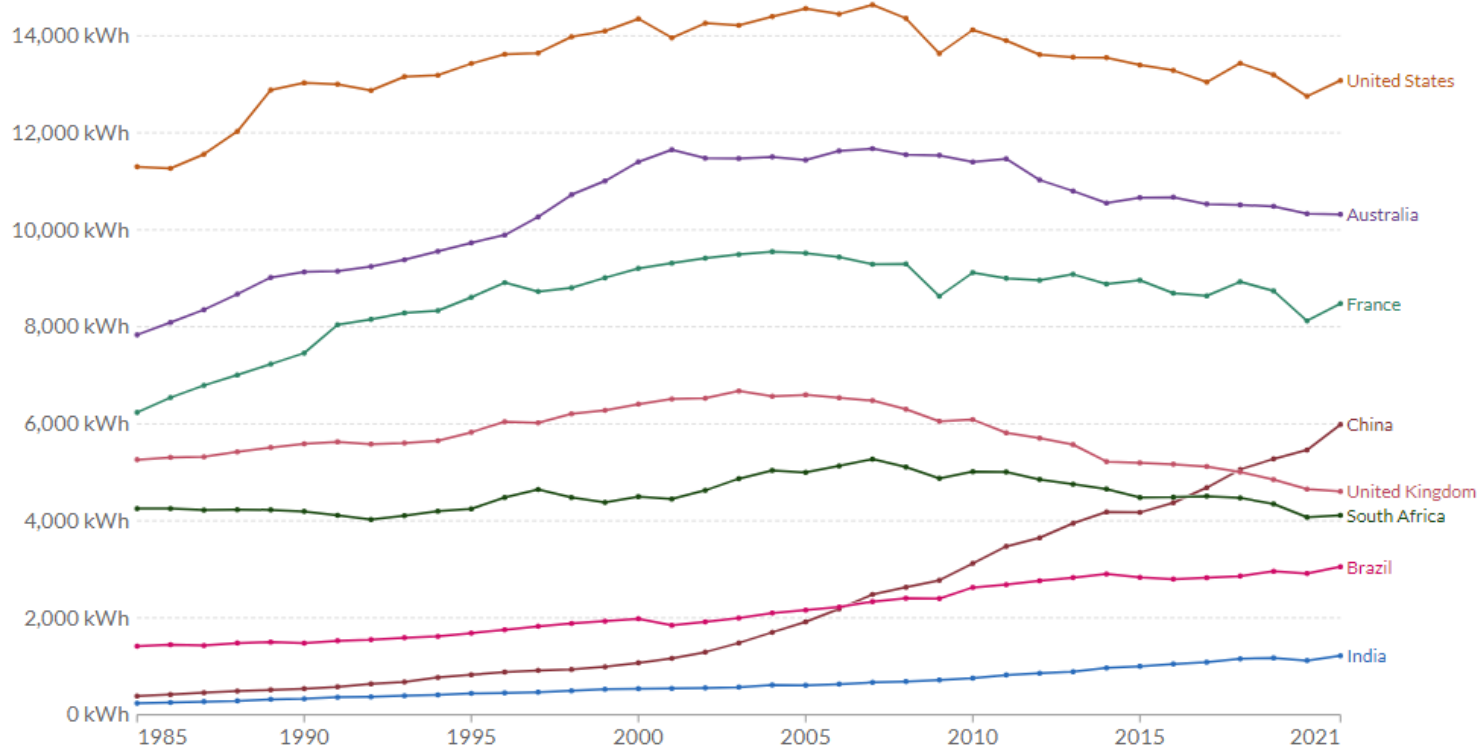


## Per capita electricity generation

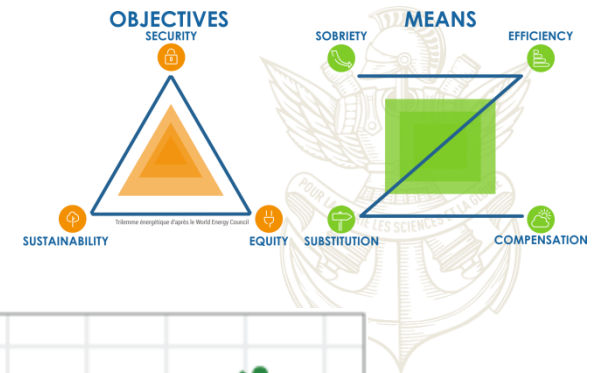
This is annual average electricity generation per person, measured in kilowatt-hours.



[+ Add country](#)



# Efficiency - transport



How far can you go with 1L of gasoline ?

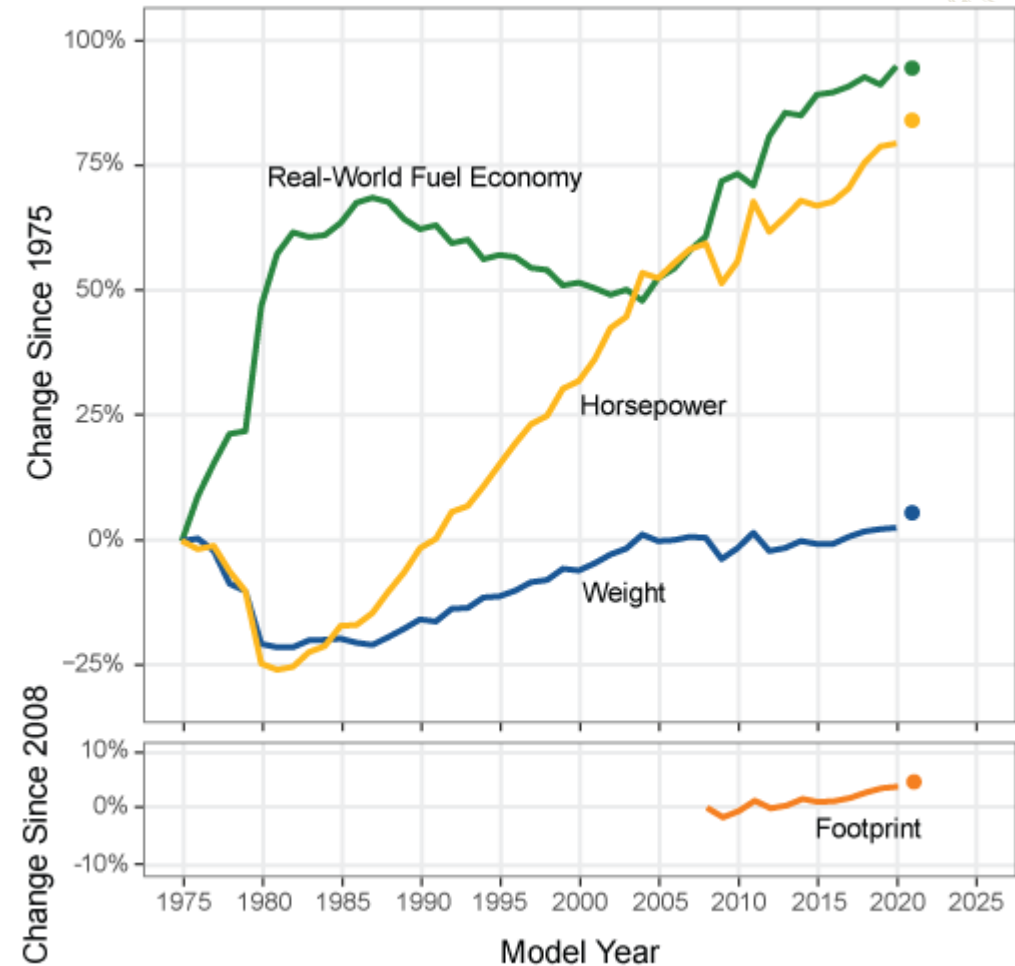


3800 km / L !

Individual transport technology ?

Individual transport behavior (car sharing) ?

Public transport ?

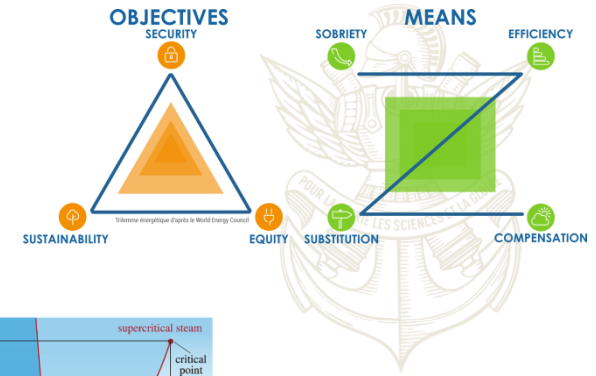
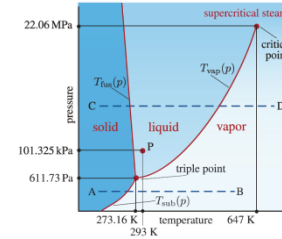
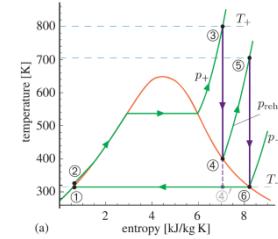
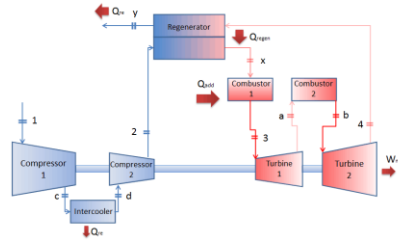


Data : EPA

# Efficiency - power

Improving technologies

Production



Distribution

See lecture 9

Demand

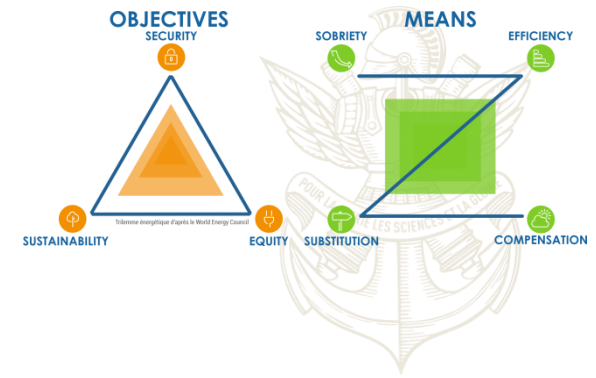
Efficient electronics...



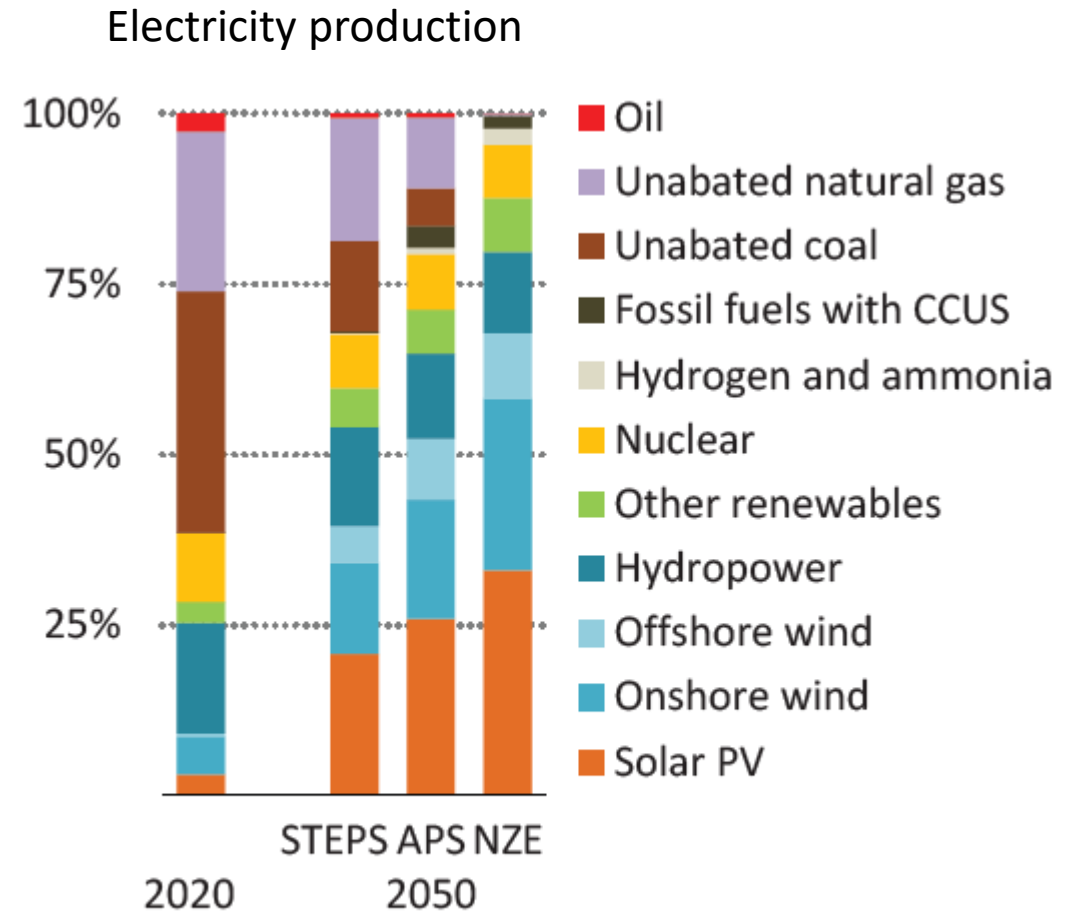
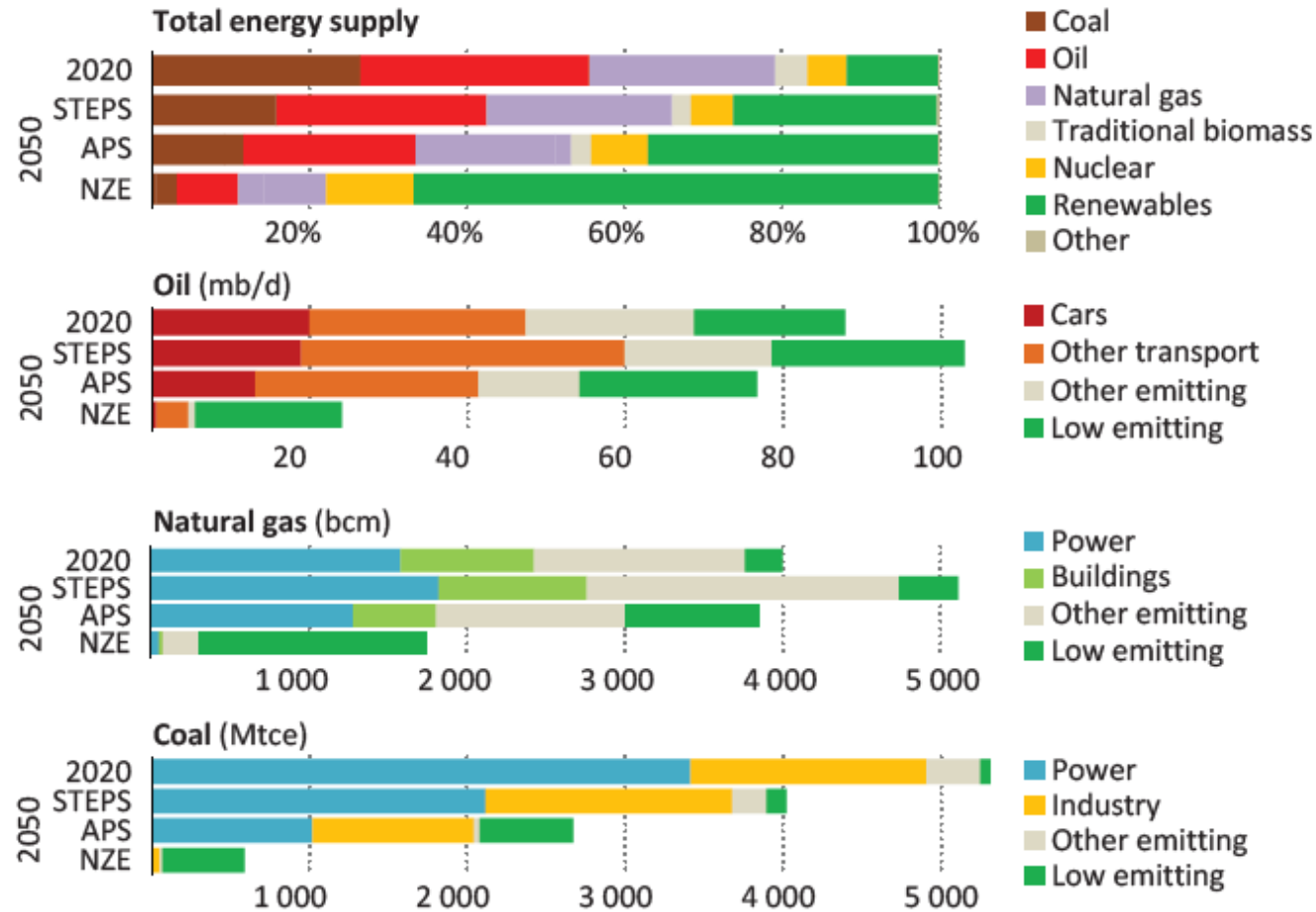
Improving usage

Combined cycle, cogeneration

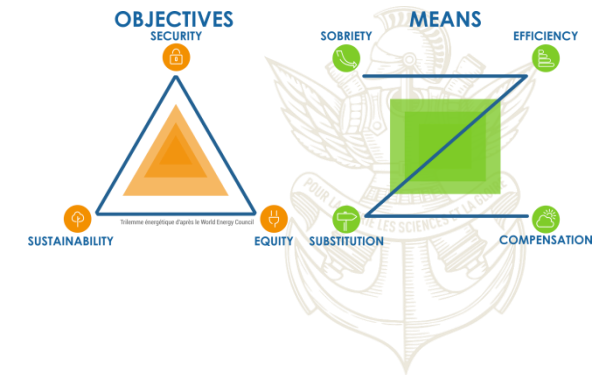
# Substitution - combustion



**Figure 4.15** ▶ Energy supply and demand by fuel and sector, 2020 and 2050



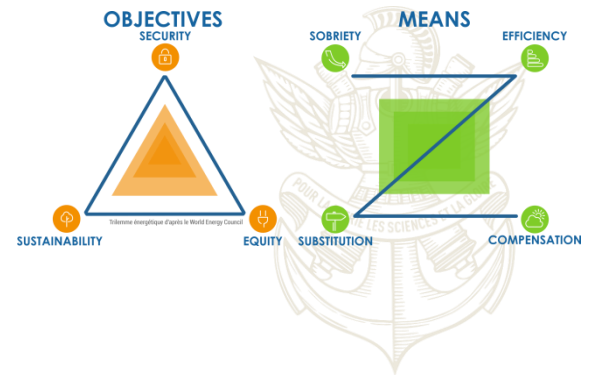
# Substitution – direct conversion



## Gradient

		Displacement gradient (Solid strain)	Hydraulic gradient	Chemical gradient	Electrical gradient	Thermal gradient
	Solid stress	<b>Hooke's law</b>	Effective stress principle	Adsorption-induced stress	Piezoelectric effect	Thermal stress
	Fluid flow	Skempton's effect	<b>Darcy's law</b>	Chemo-osmosis	Electro-osmosis	Thermo-osmosis
Flux	Species transport	Strain-induced adsorption	Streaming current	<b>Fick's law</b>	Electrophoresis	Soret effect
	Electric current	Piezoelectric effect	Streaming potential	Diffusive current	<b>Ohm's law</b>	Seebeck effect
	Heat transfer	Coupled thermoelasticity	Thermal filtration	Dufour effect	Peltier effect	<b>Fourier's law</b>

# Compensation



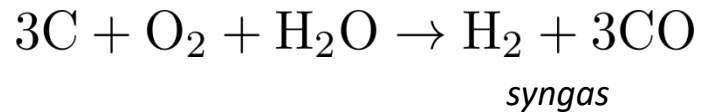
« no unabated coal » : still coal, but « clean » coal ?

Work required to get 1 mole of CO<sub>2</sub> from a mixture with concentration x

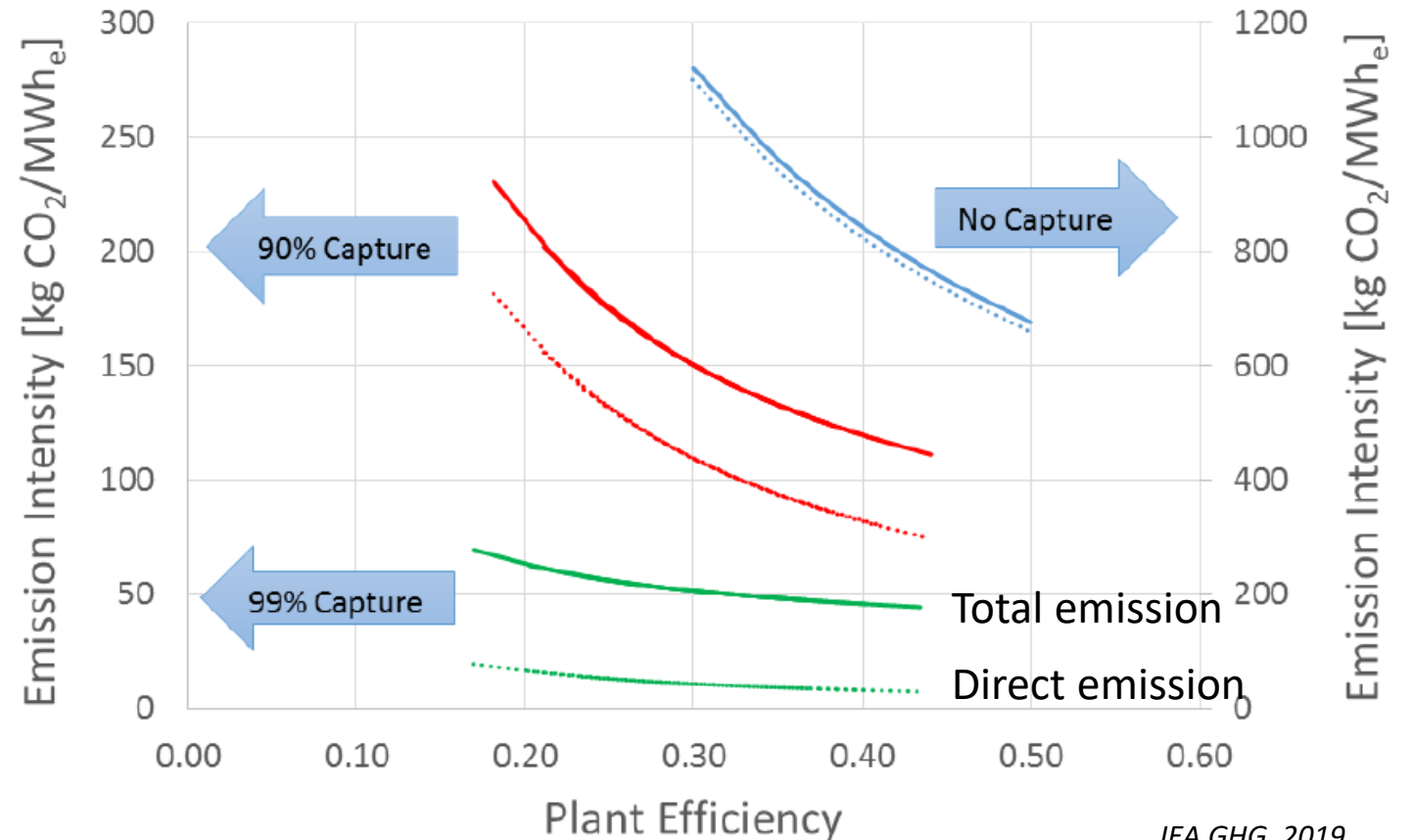
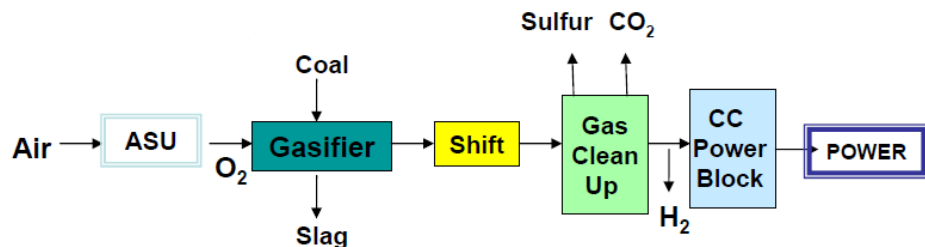
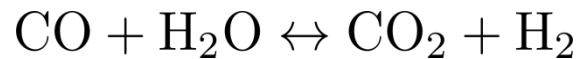
$$W = RT(1 - \ln x)$$

Concentrate even more CO<sub>2</sub> ?

Integrated gasification combined cycle



water-gas shift reaction

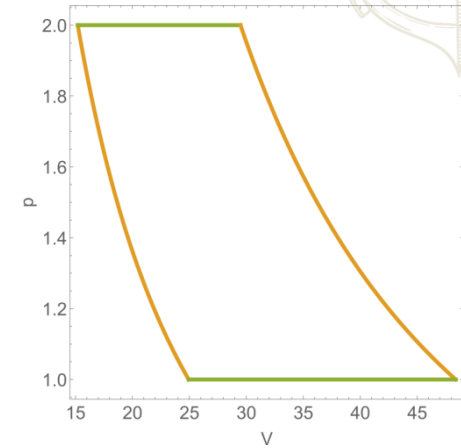
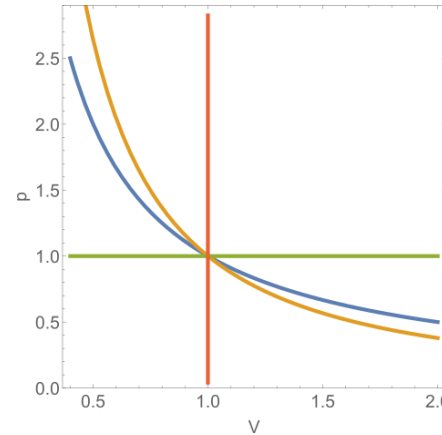


# Take home message

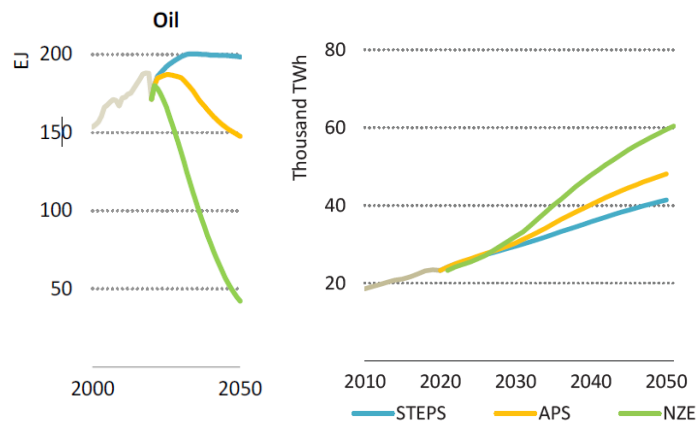


$$dU = \delta W + \delta Q$$

$$dS = \frac{\delta Q}{T_{\text{ext}}} + \delta S_{\text{cr}}$$



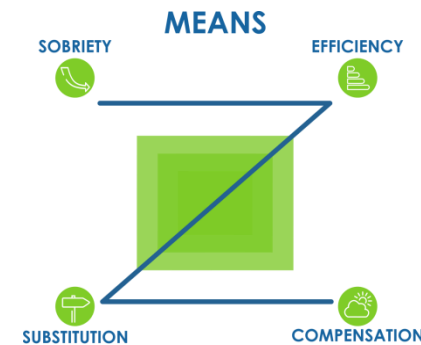
Heat, work  
Conversion efficiency  
Power



Ideal gas properties  
Basic diagrams

4 standard cycles  
Specificities, usage...

Perspective



Solutions